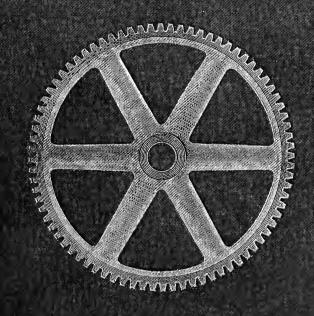
FORMULAS IN GEARING

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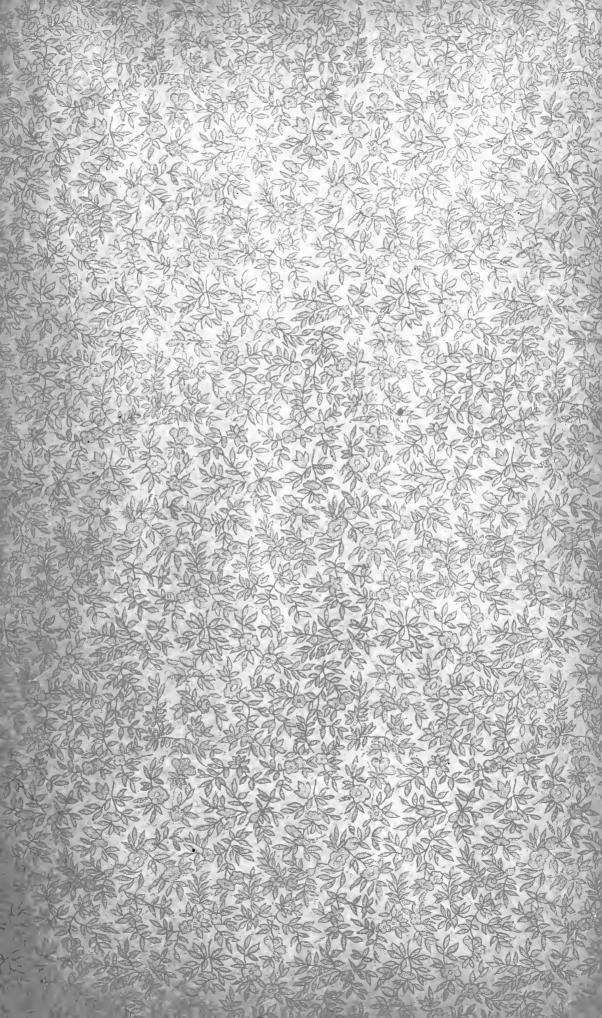


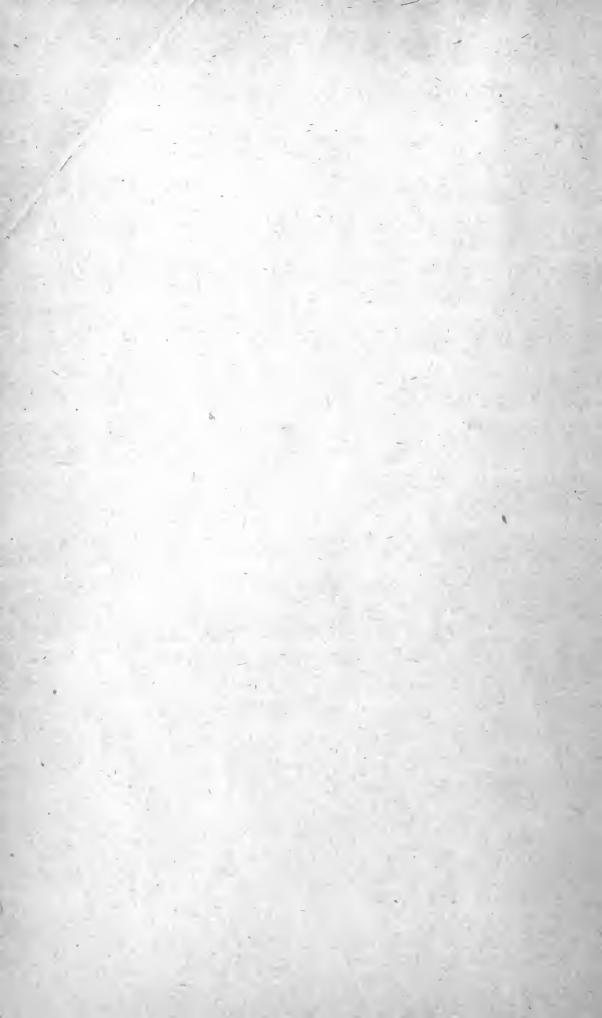
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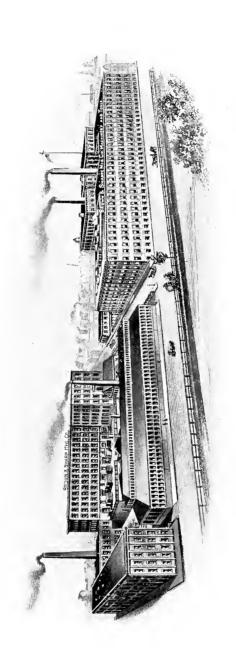












FORMULAS

IN

GEARING.

THIRD EDITION.

WITH PRACTICAL SUGGESTIONS.

PROVIDENCE, R. I.

BROWN & SHARPE MANUFACTURING COMPANY

1900.

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PREFACE.

It is the aim, in the following pages, to condense as much as possible the solution of all problems in gearing which in the ordinary practice may be met with, to the exclusion of problems dealing with transmission of power and strength of gearing. The simplest and briefest being the symbolical expression, it has, whenever available, been resorted to. The mathematics employed are of a simple kind, and will present no difficulty to anyone familiar with ordinary Algebra and the elements of Trigonometry.

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FORMULAS IN GEARING.

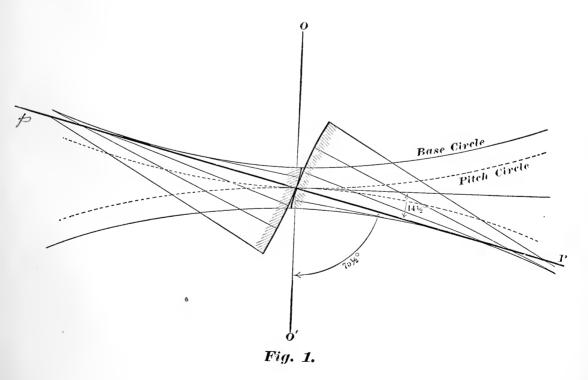
CHAPTER I.

SYSTEMS OF GEARING.

(Figs. 1, 2.)

There are in common use two systems of gearing, viz.: the involute and the epicycloidal.

In the involute system the outlines of the working parts of a tooth are single curves, which may be traced by a point in a flexible, inextensible cord being unwound from a circular disk the circumference of which is called the base circle, the disk being concentric with the pitch circle of the gear.

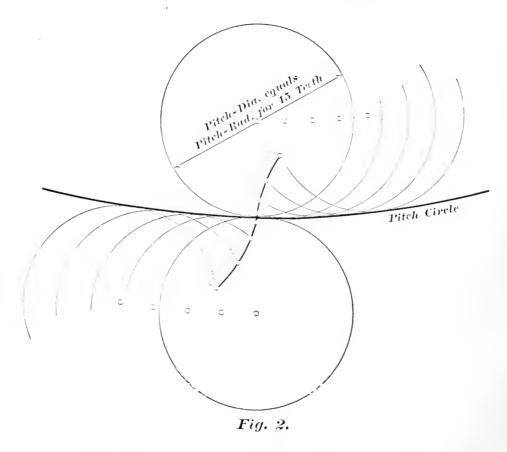


In Fig. 1 the two base circles are represented as tangent to the line PP. This line (PP) is variously called "the line of pressure," "the line of contact," or "the line of action."

In our practice this is drawn so as to make with a normal to the center line $(O O') 14\frac{1}{2}^{\circ}$, or with the center line $75\frac{1}{2}^{\circ}$.

The rack of this system has teeth with straight sides, the two sides of a tooth making, together, an angle of 29° (twice $14^{1}2^{\circ}$).

This applies to gears having 30 teeth or more. For gears having less than 30 teeth special rules are followed, which are explained in our "Practical Treatise on Gearing."



In epicycloidal, or double-curve teeth, the formation of the curve changes at the pitch circle. The outline of the faces of epicycloidal teeth may be traced by a point in a circle rolling on the outside of pitch circle of a gear, and the flanks by a point in a circle rolling on the inside of the pitch circle. The faces of one gear must be traced by the same circle that traces the flanks of the engaging gear.

In our practice the diameter of the rolling or describing circle is equal to the radius of a 15-tooth gear of the pitch required; this is the base of the system. The same describing circle being used for all gears of the same pitch.

The teeth of the rack of this system have double curves, which may be traced by the base circle rolling alternately on each side of the pitch line.

An advantage of the involute over the epicycloidal tooth is, that in action gears having involute teeth may be separated a little from their normal positions without interfering with the angular velocity, which is not possible in any other kind of tooth.

The obliquity of action is sometimes urged as an objection to involute teeth, but a full consideration of the subject will show that the importance of this has been greatly over-estimated.

The tooth dimensions for both the involute and epicycloidal gears may be calculated from the formulas in Chapter II.

CHAPTER II.

SPUR GEARING.

(Figs. 3, 4.)

Two spur gears in action are comparable to two corresponding plain rollers whose surfaces are in contact, these surfaces representing the pitch circles of the gears.

PITCH OF GEARS.

For convenience of expression the pitch of gears may be stated as follows:

Circular pitch is the distance from the center of one tooth to the center of the next tooth, measured on the pitch line.

Diametral pitch is the number of teeth in a gear per inch of pitch diameter. That is, a gear that has, say, six teeth for each inch in pitch diameter is six diametral pitch, or, as the expression is universally abbreviated, it is "six pitch." This is by far the most convenient way of expressing the relation of diameter to number of teeth.

Module is the pitch diameter of a gear divided by the number of teeth.

Chordal pitch is a term but little employed. It is the distance from center to center of two adjacent teeth measured in a straight line.



Fig. 3.

FORMULAS.

N = number of teeth.

s = addendum and module.

t =thickness of tooth on pitch line.

f = clearance at bottom of tooth.

D'' =working depth of tooth.

D'' + f =whole depth of tooth.

d = pitch diameter.

d' = outside diameter.

P' = circular pitch.

 $P^c = \text{chord pitch.}$

P = diametral pitch.

C = center distance.

$$P = \frac{N+2}{d'}$$

$$P = \frac{\pi}{P'}$$

$$P' = \frac{\pi}{P}$$

$$s = \frac{I}{P} = \frac{P'}{\pi} = .3183 P'$$

$$s = \frac{d}{N} = \frac{d'}{N+2}$$

$$t = \frac{I}{2} P' = \frac{\pi}{2P}$$

$$f = \frac{I}{10} t$$

$$s + f = \frac{I}{P} \left(I + \frac{\pi}{20} \right) = .3685 P'$$

$$D'' = 2 s$$

$$P^{c} = d \sin \frac{180^{\circ}}{N}$$

$$P' = d \pi \frac{\delta}{360^{\circ}} \text{ where } \sin \delta = \frac{P^{c}}{d}$$

$$d = \frac{N}{P}$$

$$d' = d + 2 s$$

$$d = \frac{N P'}{\pi}$$

GEAR WHEELS.

TABLE OF TOOTH PARTS—CIRCULAR PITCH IN FIRST COLUMN.

Gircular Pitch.	Threads or Teeth per inch Linear.	Diametral Pitch.	Thickness of Tooth on Pitch Line.	Addendum and Module.	Working Depth of Tooth.	Depth of Space below Pitch Line.	Whole Depth of Tooth.	Width of Thread-Tool at End.	Width of Thread at Top.
P	1" P'	Р	t	S	$D^{''}$	s+f	D'' + f	P [′] X.31	P'X.335
2	1 2	1.5708	1.0000	.6366	1.2732	.7366	1.3732	.6200	.6700
$1\frac{7}{8}$	<u>8</u> 15	1.6755	.9375	.5968	1.1937	.6906	1.2874	.5813	.6281
$1\frac{3}{4}$	<u>+</u> 7	1.7952	.8750	.5570	1.1141	.6445	1.2016	.5425	.5863
$1\frac{5}{8}$	8 13	1.9333	.8125	.5173	1.0345	.5985	1.1158	.5038	.5444
$1\frac{1}{2}$	$\frac{2}{3}$	2.0944	.7500	.4775	.9549	.5525	1.0299	.4650	.5025
$1\frac{7}{16}$	16 23	2.1855	.7187	.4576	.9151	.5294	.9870	.4456	.4816
$1\frac{3}{8}$	<u>8</u> 11	2.2848	.6875	.4377	.8754	.5064	.9441	.4262	.4606
$1\frac{1}{3}$	3 4	2.3562	.6666	.4244	.8488	.4910	.9154	.4133	.4466
$1\frac{5}{16}$	16 21	2.3936	.6562	.4178	.8356	.4834	.9012	.4069	.4397
$1\frac{1}{4}$	<u>4</u> 5	2.5133	.6250	.3979	.7958	.4604	.8583	.3875	.4188
$1\frac{3}{16}$	16 19	2.6456	.5937	.3780	.7560	.4374	.8156	.3681	.3978
1 1/8	8 9	2.7925	.5625	.3581	.7162	.4143	.7724	.3488	.3769
$1\frac{1}{16}$	16 17	2.9568	.5312	.3382	.6764	.3913	.7295	.3294	.3559
1	1	3.1416	.5000	.3183	.6366	.3683	.6866	.3100	.3350
$\frac{15}{16}$	$1\frac{1}{15}$	3.3510	.4687	.2984	.5968	.3453	.6437	.2906	.3141
7 8	$1^{\frac{1}{7}}$	3.5904	.4375	.2785	.5570	.3223	.6007	.2713	.2931
13	$1\frac{3}{13}$	3.8666	.4062	.2586	.5173	.2993	.5579	.2519	.2722
5	$1\frac{1}{4}$	3.9270	.4000	.2546	.5092	.2946	.5492	.2480	.2680
3 4	$1\frac{1}{3}$	4.1888	.3750	.2387	.4775	.2762	.5150	.2325	.2513
11 16	$1\frac{5}{11}$	4.5696	.3437	.2189	.4377	.2532	.4720	.2131	.2303
3	$1^{\frac{1}{2}}$	4.7124	.3333	.2122	.4244	.2455	.4577	.2066	.2233
<u>5</u> 8	$1\frac{3}{5}$	5.0265	.3125	.1989	.3979	.2301	.4291	.1938	.2094
<u>3</u> 5	$1^{\frac{2}{3}}$	5.2360	.3000	.1910	.3820	.2210	.4120	.1860	.2010
4 7	$1\frac{3}{4}$	5.4978	.2857	.1819	.3638	.2105	.3923	.1771	.1914
16	$1^{\frac{7}{9}}$	5.5851	.2812	.1790	.3581	.2071	.3862	.1744	.1884

${\bf TABLE\ OF\ TOOTH\ PARTS.} \color{red} - Continued.$

CIRCULAR PITCH IN FIRST COLUMN.

Circular Pitch.	Threads or Teeth per inch Linear.	Diametral Pitch.	Thickness of Tooth on Pitch Line.	Addendum and Module.	Working Depth of Tooth.	Depth of Space below Pitch Line.	Whole Depth of Tooth.	Width of Thread-Tool at End.	Width of Thread at Top.
P'	1" P'	P	t	S 39	D''	s+f	D''+f.	P×.31	P×.335
1 2	2	6.2832	.2500	.1592	.3183	.1842	.3433	.1550	.1675
<u> </u>	$2\frac{1}{4}$	7.0685	.2222	.1415	.2830	.1637	.3052	.1378	.1489
7	$2\frac{2}{7}$	7.1808	.2187	.1393	.2785	.1611	.3003	.1356	.1466
3	$2\frac{1}{3}$	7.3304	.2143	.1364	.2728	.1578	.2942	.1328	.1436
<u>2</u> 5	$2\frac{1}{2}$	7.8540	.2000	.1273	.2546	.1473	.2746	.1240	.1340
3 8	$2\frac{2}{3}$	8.3776	.1875	.1194	.2387	.1381	.2575	.1163	.1256
11	$2\frac{3}{4}$	8.6394	.1818	.1158	.2313	.1340	.2498	.1127	.1218
1/3	3	9.4248	.1666	.1061	.2122	.1228	.2289	.1033	.1117
5 16	$3\frac{1}{5}$	10.0531	.1562	.0995	.1989	.1151	.2146	.0969	.1047
3 10	$3\frac{1}{3}$	10.4719	.1500	.0955	.1910	.1105	.2060	.0930	.1005
2 7	$3\frac{1}{2}$	10.9956	.1429	.0909	.1819	.1052	.1962	.0886	.0957
1/4	4	12.5664	.1250	.0796	.1591	.0921	.1716	.0775	.0838
2 9	$4\frac{1}{2}$	14.1372	.1111	.0707	.1415	.0818	.1526	.0689	.0744
1 5	5	15.7080	.1000	.0637	.1273	.0737	.1373	.0620	.0670
$\frac{3}{16}$	$5\frac{1}{3}$	16.7552	.0937	.0597	.1194	.0690	.1287	.0581	.0628
$\frac{2}{11}$	$5\frac{1}{2}$	17.2788	.0909	.0579	.1158	.0670	.1249	.0564	.0609
1 6	6	18.8496	.0833	.0531	.1061	.0614	.1144	.0517	.0558
$\frac{2}{13}$	$6\frac{1}{2}$	20.4203	.0769	.0489	.0978	.0566	.1055	.0477	.0515
$\frac{1}{7}$	7	21.9911	.0714	.0455	.0910	.0526	.0981	.0443	.0479
2 15	$7\frac{1}{2}$	23.5619	.0666	.0425	.0850	.0492	.0917	.0414	.0446
1/8	8	25.1327	.0625	.0398	.0796	.0460	.0858	.0388	.0419
1 9	9	28.2743	.0555	.0354	.0707	.0409	.0763	.0344	.0372
1 10	10	31.4159	.0500	.0318	.0637	.0368	.0687	.0310	.0335
1 16	16	50.2655	.0312	.0199	.0398	.0230	.0429	.0194	.0209
1 20	20	62.8318	.0250	.0159	.0318	.0184	.0343	.0155	.0167

GEAR WHEELS.

TABLE OF TOOTH PARTS—DIAMETRAL PITCH IN FIRST COLUMN.

Diametral Pitch.	Circular Pitch.	Thickness of Tooth on Pitch Line.	Addendum and Module.	Working Depth of Tooth.	Depth of Space below Pitch Line.	Whole Depth of Tooth.
P	P'	t	8	D''	s+f.	D''+f.
$\frac{1}{2}$	6.2832	3.1416	2.0000	4.0000	2.3142	4.3142
<u>3</u> 4	4.1888	2,0944	1.3333	2.6666	1.5428	2.8761
1	3.1416	1.5708	1.0000	2.0000	1.1571	2.1571
$1\frac{1}{4}$	2.5133	1.2566	.8000	1.6000	.9257	1.7257
$1\frac{1}{2}$	2.0944	1.0472	.6666	1.3333	.7714	1.4381
$1\frac{3}{4}$	1.7952	.8976	.5714	1.1429	.6612	1.2326
2	1.5708	.7854	. 5000	1.0000	.5785	1.0785
$2\frac{1}{4}$	1.3963	.6981	.4444	.8888	.5143	.9587
$2\frac{1}{2}$	1.2566	.6283	.4000	.8000	.4628	.8628
234	1.1424	.5712	. 3636	.7273	.4208	.7814
3	1.0472	.5236	.3333	.6666	.3857	.7190
$3\frac{1}{2}$.8976	.4488	.2857	. 5714	.3306	.6163
4	.7854	.3927	.2500	.5000	.2893	. 5393
5	.6283	.3142	.2000	.4000	. 2314	.4314
G	. 5236	.2618	.1666	.3333	.1928	.3595
7	.4488	.2244	.1429	. 2857	.1653	.3081
8	.3927	.1963	.1250	.2500	.1446	.2696
9	.3491	.1745	.1111	.2222	.1286	.2397
10	.3142	.1571	.1000	. 2000	.1157	.2157
11	.2856	.1428	.0909	.1818	.1052	.1961
12	.2618	.1309	. 0833	.1666	.0964	.1798
13	.2417	.1208	.0769	.1538	.0890	.1659
14	. 2244	.1122	.0714	.1429	.0826	.1541

TABLE OF TOOTH PARTS—Continued.

DIAMETRAL PITCH IN FIRST COLUMN.

Diametral Pitch.	Circular Pitch.	Thickness of Tooth on Pitch Line.	Addendum and Module.	Working Depth of Tooth.	Depth of Space below Pitch Line.	Whole Depth of Tooth.
P.	P'.	t.	8.	D''.	s+f.	D''+f.
15	.2094	.1047	.0666	.1333	.0771	.1438
16	.1963	.0982	.0625	.1250	.0723	.1348
17	.1848	.0924	.0598	.1176	.0681	.1269
18	.1745	.0873	.0555	.1111	.0643	.1198
19	.1653	.0827	.0526	.1053	. 0609	.1135
20	.1571	.0785	.0500	.1000	.0579	.1079
22	.1428	.0714	.0455	.0909	.0526	.0980
24	. 1309	.0654	.0417	.0833	.0482	.0898
26	.1208	.0604	.0385	.0769	. 0445	.0829
28	.1122	.0561	.0357	.0714	.0413	.0770
30	.1047	.0524	.0333	.0666	.0386	.0719
32	.0982	.0491	.0312	.0625	.0362	.0674
34	.0924	.0462	.0294	.0588	.0340	.0634
36	.0873	. 0436	.0278	.0555	.0321	.0599
38	.0827	.0413	. 0263	.0526	.0304	.0568
40	.0785	.0393	.0250	.0500	.0289	.0539
42	.0748	.0374	.0238	.0476	.0275	.0514
44 .	.0714	.0357	.0227	.0455	.0263	.0490
46	.0683	. 0341	.0217	.0435	.0252	.0469
48	.0654	.0327	.0208	.0417	.0241	.0449
50	.0628	.0314	.0200	.0400	.0231	.0431
56	.0561	.0280	.0178	.0357	.0207	.0385
60	.0524	.0262	.0166	.0333	.0193	.0360

Comparative Sizes of Gear Teeth. Involute.

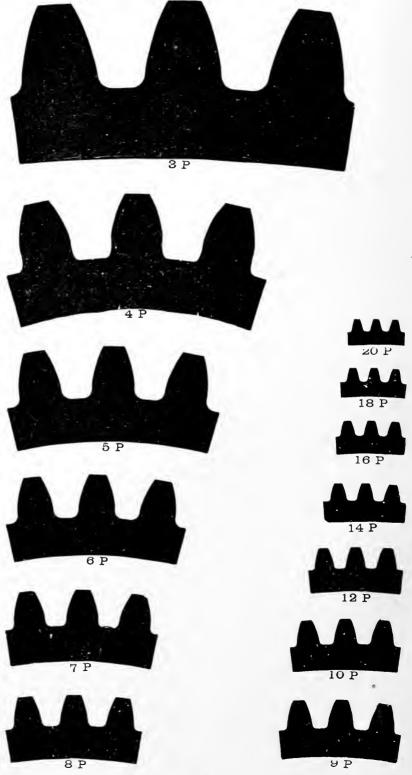
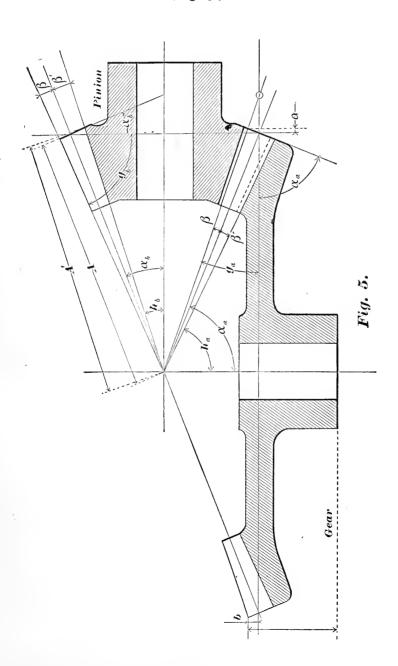


Fig. 4.

CHAPTER III.

BEVEL GEARS.—AXES AT RIGHT ANGLES.

(Fig. 5.)



FORMULAS.

 $N_a = N_b = Number of teeth \begin{cases} gear. \\ N_b = Number of teeth \end{cases}$ gear. $N_b = Number of teeth \begin{cases} gear. \\ P' = circular pitch. \end{cases}$ P' = circular pitch. P' = circular pitch. P' = circular pitch. P' = circular pitch. P' = angle of teeth angle of edge for <math>n = n gear. n = n gear or pitch angle for n = n gear. n = n gear. n = n for n = n gear. n = n for n = n for n = n for n = n gear. n = n gea

d' =outside diameter.

s = addendum and module.

t =thickness of tooth at pitch line.

f = clearance at bottom of tooth.

D'' = working depth of tooth.

D'' + f =whole depth of tooth.

a = diameter increment.

b =distance from top of tooth to plane of pitch circle.

F = width of face.

$$\tan \alpha_a = \frac{N_a}{N_b}; \quad \tan \alpha_b = \frac{N_b}{N_a};$$

$$\tan \beta = \frac{2 \sin \alpha}{N}; \text{ or } \quad \tan \beta = \frac{s}{A}.$$

$$\tan \beta' = \frac{\sin \alpha \left(2 + \frac{\pi}{10}\right)}{N} = \frac{2.314 \sin \alpha}{N}; \quad \tan \beta' = \frac{s + f}{A};$$

$$g_a = 90^\circ - (\alpha_a + \beta); g_b = 90^\circ - (\alpha_b + \beta)$$

$$h = \alpha - \beta' \quad (See \ Note, page 69.)$$

$$A = \sqrt{\frac{N_a}{2P}}^2 + \frac{N_b}{2P}^2$$

$$A = \frac{N}{2 \ P \sin \alpha}$$

$$A' = \frac{A}{\cos \beta'} \qquad A' = \frac{N}{2 \ P \sin \alpha \cos \beta'}$$

$$A = \frac{\frac{1}{2} \frac{d'}{\alpha'}}{\sin (\alpha + \beta)} \cos \beta$$

$$P = \frac{N}{2 \ A \sin \alpha}$$

$$d = \frac{N}{P} \text{ or } = \frac{N \ P'}{\pi} \qquad d' = d + 2 \ a$$

$$2 \ a = 2 \ s \cos \alpha \qquad (See \ page 20.)$$

$$b = a \tan \alpha \quad \begin{cases} a \ for \ gear = b \ for \ pinion = b \ for \ gear \end{cases}$$

$$P = \frac{\pi}{P'} \qquad P' = \frac{\pi}{P}$$

$$s = \frac{1}{P} = \frac{P'}{\pi} = .3183 \ P' \qquad s = A \tan \beta$$

$$s + f = .3685 \ P' \qquad s + f = A \tan \beta'$$

$$s + f = \frac{1}{2} \left(1 + \frac{\pi}{20}\right) \qquad D'' = 2 \ s$$

$$t = \frac{P'}{2} = \frac{\pi}{2P} \qquad f = \frac{1}{10} \ t$$

$$F = \frac{4}{P} + \frac{A}{7} \text{ or } = 2 \ P' \text{ to } 3 \ P'$$

NOTE.—Formulas containing notations without the designating letters a and b apply equally to either gear or pinion. If wanted for one or the other, the respective letters are simply attached.

BEVEL GEARS WITH AXES AT ANY ANGLE.

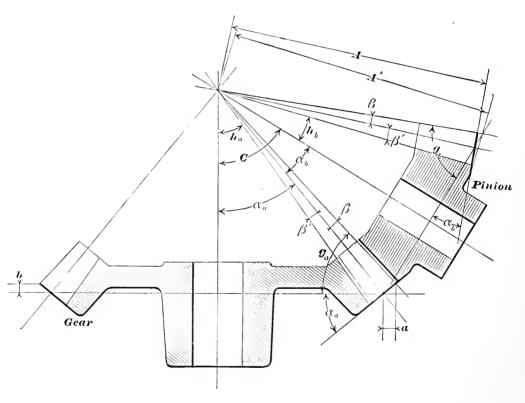


Fig. 6.

FORMULAS.

C =angle formed by axes of gears.

P = diametral pitch.

P' = circular pitch.

 $\alpha_a = \begin{cases}
\alpha_b = \end{cases}$ angle of edge = pitch angle $\begin{cases}
\text{gear.} \\
\text{pinion.}
\end{cases}$

 β = angle of top.

 β' = angle of bottom.

 $g_a = \begin{cases} g_a = \\ g_b = \end{cases}$ angle of face $\begin{cases} \text{gear.} \\ \text{pinion.} \end{cases}$

A = apex distance from pitch circle.

 $\Lambda' = \text{apex distance from large bottom of tooth.}$

d = pitch diameter.

d' = outside diameter.

a = diameter increment.

b = distance from top of tooth to plane of pitch circle.

NOTE.—The formulas for tooth parts as given on page 5 apply equally to these cases.

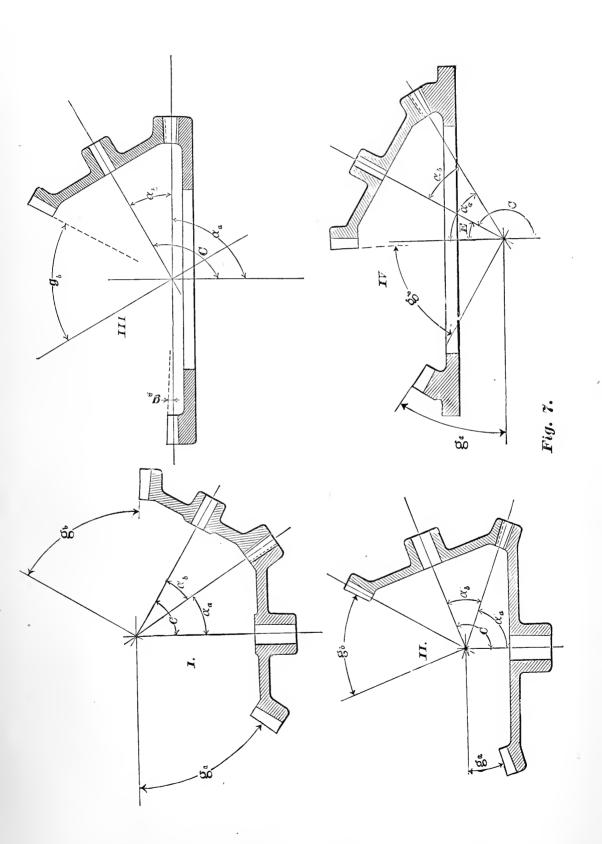
$$\tan \alpha_a = \frac{\sin C}{\frac{N_b}{N_a} + \cos C}; \text{ or } \cot \alpha_a = \frac{N_b}{N_a \sin C} + \cot C$$

$$\tan \alpha_b = \frac{\sin C}{\frac{N_a}{N_b} + \cos C}; \text{ or } \cot \alpha_b = \frac{N_a}{N_b \sin C} + \cot C$$

Note.—The above formulas are correct only for values of C less than 90°. If C is greater than 90°, consult page 18.

$$\tan \beta = \frac{2 \sin \alpha}{N}$$
; or $\tan \beta = \frac{s}{A}$;
 $\tan \beta' = \frac{\sin \alpha \left(2 + \frac{\pi}{10}\right)}{N} = \frac{2.314 \sin \alpha}{N}$; $\tan \beta' = \frac{s+f}{A}$;
 $g_{\alpha} = 90^{\circ} - (\alpha_{\alpha} + \beta)$ for Cases I and II.
 $g_{\alpha} = \beta$, for Case III.
 $g_{\alpha} = 90^{\circ} - (\alpha_{\alpha} - \beta)$ for Case IV.
 $g_{b} = 90^{\circ} - (\alpha_{b} + \beta)$
 $h = \alpha - \beta'$ (See page 69.)
 $A = \frac{N}{2 \text{ P sin } \alpha}$
 $A' = \frac{A}{\cos \beta'}$
 $d = \frac{N}{p} \text{ or } = \frac{N}{\pi}$
 $d' = d + 2 a$ (for Cases I and II.
 $d' = d + 2 a$, for gear in Case III.
 $d' = d - 2 a$, for gear in Case IV.
 $2 a = s \cos \alpha$
 $b = s \sin \alpha$

Note.—Formulas containing notations without the designating letters a and b apply equally to either gear or pinion. If wanted for one or the other, the respective letters are simply attached.



The formulas given for α_a and α_b (when C. N_a and N_b are known) undergo some modifications for values of C greater than 90°.

For bevel gears at any angle but 90° we may distinguish four cases; C, N_a , N_b being given.

I. Case. See pages 14 and 16.

II. Case. C is greater than 90°.

$$\tan \alpha_a = \frac{\sin (180 - C)}{\frac{N_b}{N_a} - \cos (180 - C)}; \quad \tan \alpha_b = \frac{\sin (180 - C)}{\frac{N_a}{N_b} - \cos (180 - C)}$$

III. Case. $\alpha_a = 90^\circ$; $\alpha_b = C - 90^\circ$

IV. Case.

$$\tan \alpha_{a} = \frac{\sin E}{\cos E - \frac{N_{b}}{\bar{N}_{a}}}; \quad \tan \alpha_{b} = \frac{\sin E}{\frac{N_{a}}{\bar{N}_{b}} - \cos E}$$

For an example to apply to Case III., the following condition must be fulfilled:

$$N_a \sin (C - 90^\circ) = N_b$$

To distinguish whether a given example belongs to Case II. or case IV., we are guided by the following condition:

Is: $N_a \sin (C - 90^\circ) \begin{cases} smaller \text{ than } N_b, \text{ we have Case II.} \\ larger \text{ than } N_b, \text{ we have case IV.} \end{cases}$

UNDERCUT IN BEVEL GEARS.

By undercut in gears is understood a special formation of the tooth, which may be explained by saying that the elements of the tooth below the pitch line are nearer the center line of the tooth than those on the pitch line. Such a tooth outline is to be found only in gears with few teeth. In a pair of bevel gears where the pinion is low-numbered and the ratio high, we are apt to have undercut. For a pair of running gears this condition presents no objection. Should, however, these gears be intended as patterns to cast from, they would be found useless, from the fact that they would not draw out of the sand. We have stated on page 2 (see Fig. 1) that the base of our involute system is the 14½° pressure angle. If a pair of bevel gears with teeth constructed on this basis have undercut, we can nearly eliminate the undercut—and for the practical working this is quite sufficient-by taking as a basis for the construction of the tooth outline a pressure angle of 20°.

The question now is: When do we, and when do we not have undercut? Let there be:

N = number of teeth in gear. n = number of teeth in pinion.

$$\frac{n\sqrt{N^2+n^2}}{N} = p$$

where we have undercut for p less than 30.

This formula is strictly correct for epicycloidal gears only. It is, however, used as a safe and efficient approximation for the involute system.

DIAMETER INCREMENT.

2 a.

Rule.—The ratio being given or determined, to find the outside diameter divide figures given in table for large and small gear by pitch (P) and add quotient to pitch diameter.

RATIO.	GE	GEARS.		RATIO.		ARS.	RAT	RATIO.		GEARS.	
	Large	Small			Large	Small			Large	Small	
1.00 1:1 1.05 1.07 1.10 10:9 1.11 1.12 1.13 9:8 1.14 8:7 1.15 1.16 1.17 7:6 1.18 1.19 1.20 6:5 1.23 1.25 5:4 1.27 1.29 9:7 1.30 4:3 1.35 4:3	1.41 1.37 1.36 1.35 1.34 1.33 1.33 1.32 1.31 1.30 1.29 1.28 1.27 1.25 1.24 1.22 1.20 1.18	1.41 1.42 1.43 1.44 1.46 1.47 1.49 1.50 1.51 1.52 1.53 1.55 1.55 1.56 1.57 1.58 1.59 1.60 1.61	1.65 1.67 1.70 1.75 1.80 1.85 1.90 1.95 2.00 2.20 2.25 2.30 2.33 2.40 2.50 2.67 2.70 2.80 2.90	5:3 7:4 9:5 2:1 9:4 7:3 5:2 8:3	1.05 1.03 1.01 .99 .97 .95 .93 .91 .89 .87 .84 .82 .76 .75 .71 .69 .67	1.70 1.72 1.73 1.74 1.75 1.76 1.77 1.78 1.79 1.80 1.81 1.82 1.83 1.84 1.85 1.86 1.87 1.87	4.40 4.50 4.60 4.80 5.00 5.20 5.40 5.60 5.80 6.20 6.40 6.60 7.20 7.40 7.60 7.80 8.00 8.20	9:2 5:1 6:1 7:1 8:1	.45 .44 .42 .41 .39 .38 .37 .36 .34 .33 .32 .31 .30 .29 .28 .27 .26 .26 .25 .24	1.94 1.95 1.96 1.96 1.96 1.96 1.97 1.97 1.97 1.97 1.97 1.98 1.98 1.98 1.98 1.98	
$\begin{array}{c cccc} 1 & 37 \\ 1 & 40 \\ 1 & 43 \\ 1 & 45 \\ 1 & 50 \\ 1 & 53 \\ 1 & 55 \\ 1 & 58 \\ 1 & 60 \\ \end{array}$ $\begin{array}{c cccc} 7:5 \\ 10:7 \\ 3:2 \\ 3:2 \\ 8:5 \\ 1 & 60 \\ \end{array}$	1.15 1.13 1.11 1.10 1.09 1.08	1.61 1.62 1.63 1.65 1.66 1.67 1.68 1.68	3.00 3.20 3.33 3.40 3.50 3.60 3.80 4.00 4.20	7:2 4:1	.63 .60 .58 .56 .54 .52 .50 .49 .47	1.91 1.92 1.92 1.93 1.93 1.94 1.94 1.94	8.40 8.60 8.80 9.00 9.20 9.40 9.60 9.80 10.00	9:1 10:1	.24 .23 .23 .22 .22 .21 .21 .20 .20	1.98 1.98 1.98 1.99 1.99 1.99 2.00 2.00 2.00	

NOTE.—To be used only for bevel gears with axes at right angle.

TABLES FOR ANGLES OF EDGE AND ANGLES OF FACE.

The following four tables have been computed for the convenience in calculating datas for bevel gears with axes at right angle. They *do not hold* good for bevel gears with axes at any other angle.

To use the tables the number of teeth in gear and pinion must be known.

Having located the number of teeth in the gear on the horizontal line of figures at the top of the table, and the number of teeth in the pinion on the vertical line of figures on the left-hand side, we follow the two columns to the square formed by their intersections.

The two angles found in the same square are the respective angles for gear and pinion. The tables are so arranged that the angle belonging to the gear is always placed above the angle for the pinion.

TABLE 1

ANGLE OF EDGE. GEAR.

13	_				37	36	35	34	33	32	31	30	29	28	27
13	16°19		72°54								•	~			-
13	16°19 72°25						71°5								66°
13		71 59	71 34	17 32 71 7	17 58 70 39	18 26 70°9	18 \$5 69°37	19 26 69°5	19 59	20 34 67 53	67 15	21 48 66 34	22 29 65 si	23 12 65 6	23's
		18°4	18 26	18°53′	19 81	19°51	20°23	20 \$5	21°36	22°7	2245	23 26	24°9′	24°54	254
		70 43 19 17													62's
	69'54	69'26	6858	68 88	67 56	67°23	6648	66 12	65 33	64 53	64 10	63.56	6239	61 48	60
		20°34													
		68°12' 21°48													
17	67°29′	66°58	66°27	65°54	65°19	64 43	64°6	63 26	62°45	62°1'	61°15'	60,68	59°37	58 44	574
		23°2′													
	66°18' 23°42'	65 46 24°14	05 H 24°46	25°21	25 56	2634	62 47 27°13	62 6 27 54	28 37	29°22	30° 9′	59 2 30°56	31°56	32 44	56°i
10	65°8′	64°36	64°2	63 26	62°49	65,10	61°30	60°48	60°4′	59°18	58°30	57°39′	56 46	55 51	54 4
_		25°24					28°30 60°15								
	64° oʻ 26° oʻ	2634													
21	62°53	62°18	61°42	61°4′	60°25	59 45	59° 2′	58 îs	57°32	56 43	55°53	55° o	54°5	53°7	52°
		2742					30°58								
	61°47 28°13	2849													
22	60°42	60°6	59°28	58°49	58°8′	57 25	56 41	55 55	55°7′	54° 18	53°26	52°31	51°35	5036	4 5 °
		29°54 59°2													
		30 58													
25	58°38	58°0′	57°20	56°40	55°57	55°13′	54 28	53 40	52°51	52°0′	51°7′	50°12	49°14	48°14	47°
—		32°0′ 56°58													
20	32 23	33°e′	33°41	34 23	35°6′	35 50	36 36	37 24	38 4	39°6′	39°59	40°55	41°53	42 53	43
		55°S													45
28	33 22 5540	34° ı′	54°19	53°37	52°53	36 53 52°8	51°20	50°32	49°41	48 48	41 3 47°55	46 58	46 0	43 58	-
														45°	
		54°3′ 35°57													
20	5348	53'7	52°26	51 42	50°s	50°12	49 24	48°35	47 43	46°51	45 56	4 =0		,	
30	36 12	36°53	37°34	38 18	39° 2	39°46	40 36	41 25	42 17	43°9	44°4′	43]		
31	52.54	52°13	51 31 48 %	50 48	39°5	49 16	48 28	47 39	4647	45 54	45				
-		51 20										,			
\rightarrow		38 40								+3	J				
		50°29 39°31													
24	50 20	49 38	48°55	48°11'	47 25	46°38	45 50	4 =0	1	,					
0.4	3940 4931	40°22	41 5 48 5	41 49	42 35 46°35	45 48	44 10		J						
33	40°29	41 12	41 55	42 39	43 25	44 12	45°								
		48°0 42°0				45°									
27	47°56	47 14	46°30	4546	1.00	-)								
3/	42°4'	42°46	43°30	44 4	45°										
38	47 10	46 28	45 45	45°											
	46 26	45 43			J										
33	48 34	44 17	173]											
40	45 42 44 18	45°													
41	45°														
71	1	١													

TABLE 1.—(Continued.)

ANGLE OF EDGE.

GEAR.

	26	25	24	23	22	21	20	19	18	17	16	15	14	13	12	
12					61 23									47°17	45°	Γ
					28°37 59°25							38 40 49°5				
13					30°35											
1 4					57°32										•	
14					32 28	_				_		43°2′	73			
15					55°43′ 34°17′							45°				
					53°58								1			
16	31°37	32°37′	33°41	34°49	36°2′	37°18′	38°46	40°6	41°38	43°16	45°					
17					52 18											
!/		54°15			37°42′ 50°43′						i					
18					39 17											
19	53°Sı′	52 46	51°38	50°26	49°11′	47°52	46°28	AE°		•						
					40 49											
20		51°20 38°40			47°43′ 42°17′		1 A.E.									
21	51°4'	49°58	48°48′	47°36	46°20	450		1								
41	38 56	40°2′	41°12	42 24	43°40	45°										
22		48°39 41°21		46°16 43°44	175											
23	48°30	47°23	46°13	ΛE°												
	41 30			<u> </u>												
24		46°10 43°50	45°													
25	46°7′ 43°53	4														
26	$\overline{}$															

$$\tan\,\alpha_a = \frac{N_a}{N_b}$$

$$\tan\,\alpha_b = \frac{N_b}{N_a}$$

(See page 13.)

TABLE 2.

Angle of Edge. GEAR.

	72	71	70	69	68	67	66	65	64	63	62	61	60	59	58	57
12	80°33 9° 2 7	80°26 9°35	80°16 9°44	80°8 9°52	79°59			79°32					78°41	78°36	78 19	
13	79°46 10°14	79°37 10°23	79 29	79°20	79°11	79°1	18°51 11°9′	78°41	78°31 11°29	78°20	78° 9′	77°58	77°46		77°22	77°9′
14	79°0	78°51	78°41	78°32	78°22	78°11	78° ı'	77°51	77°46	77 28	77°17	77°5	76°52	76°39	76°26	76 12
-	11°0	11°9	11°19	77°44	11 38 77°34		11 59 77°12	12° 9	-			12°55 76°11	13 8 75°58	13°21		_
15	11°46	11°56	12°6′ 77°7′	12° 16	12°26 76°45		-	13° oʻ 76° ió		-	13°36	13°49 75°18	14° 2'	14° 16	14 30 74°35	
16	12°32	12 42	12°53	13° 3'	13°15	13 26	13°38	13°s0	14° 2	14 15	14°28	14°42	14 56	15°เí	1 5 °25	15°41
17	76°43 13°17	76°32 13°28	76°21 13°39	13°50	75°58 14°2	14° #	75°33 14°27	14°39	14°52	15° 6	15°20	15°35			73°40 16°20	16 36
18	75°58 14° 2	75°46 14°14	75°35 14°25	75°21 14°37	75°16 14°56		74°45 15°15	74°31 15°29		74°3 15°52	73°49 16°11		73°18	73°2 16°58	72°45 17°15	
19	75°13 14°47	75°1 14°59	74°49 15°11	74°36 15°24	74° 23		73°56 16°4			73°23	72°58	72°42	72°20	72°9 17°51	71°52 18°8′	
20	74° 29 15° 31	74 16 15 44	74°3′	73°50	73°37 16°23	73°23	73°9 16°51	72°54	72°39	72°23	72°7	71°51	71°34		70°59	70°40
21	73 48	73°92	73 18	73°4	72°50	72°36	72°21	72.6	71°50	71°34	71°17	71° o	70°43	70°24	70 6	69°46
22	16°15	16 28 72 47	72°33	72 19	72°4	71°49	71°34		71° 2	70°45	70°28	70°10	19°17	69°33	19°54 69°13	68°54
22	16°59	17 13 72°3	17°27	17°41		18°11	18°26	18°42		19°15 69°57	19°34 69°39		20°8	20°27 68°42	20°47 68°22	
23	17°43 71°34	17°57	18°11′ 71°5′	18°26	18°41	18°57	19°13		19 46		20°21	20°40		21 18		21°58
24	18 %	18 41	18°55	19° 11	19°26	19043	19°59	20°16	20°34	20°51	21°10	21°29	21°48	22° 8	22°29	22°50
25	70°51 19°9	70°36 19°24	70°21 19°39	70° 5′ 19° 55				68°57 21°3			68° 3' 21° 57		67°23 22°37	67°2' 22°58	66°41' 23°19	
26	70°9′ 19°51	69°53 20°7		69°21 20°39	69°4 20°56	68°46	68°30	68°12 21°46	67°54 22° 6	67°34 22°26	67°15 22°45	66°55 23°5		66°13		65 [°] 29 24°31
27	69 27	69°10	68 54	68°38 21°22	68°20	68°3	67°45	67°26 22°34	67°8	66°48	66°28	66°7	65°46	65°25	65°2′	64°39
28	68°45	68 ² 29	68°12	67°55	67°37	67°19	67°1	66°42	66°22	66° 2	65.42	65°21	64°59	64° 37	64° 14	63°50
20	21°15 68°4′	6747	6730	67°12	66 54	66°36	66°17	23°18 65°57	65°37	65°16	64°55	64°94	64°12	63°50	63°26	63°2
20	21 56 67 23	22 13 67 6		22 48 66°30				24°3						26 10 63° 3°		26°58
30	22°37	22°54 66°25						24°46 64°30					$\overline{}$	26°57 62°18		-
31	23 18	23 35	23 54	24 12	24° 31	24 50	25°10	25°36	25°51	26° 12	26°34	26°57	27°20	27°42	28 7	28°32
32	66°2′ 23°56	65 44 24 16	24°34		25°12	25 32	25°52		26°34	26°56	27°18	27°41	28 4		28 53	60°41 29° 19
33	65°23 24°37	65 4 24 56	6445 25 15	64°26 25°34	64°7′ 25°53	63°47 26°13	63°26 26°34	63°5 26°55	62°43 27°17	62°2i 27°39	61 58 28 2	61°35 28°25	61°11 28°49	60°47 29°13		59°56 30°4
34	0.0	64°25	64°5	63°46	6326	63° s	62 45	62°23	62° 1	61°38	61°15	60°52	60°28	60° 3	59°37	
35	64° 5′	63°45	63°26	63° 6	62 46	62°25	62°4		61°19	60°57	60°33	60° 9	59°45	59°19	58°53	58°27
36	63°26	63°7′	62°47	62°27	62°6	61°45	61°23	61° 1	60°38	60°15	59°si	59°27	59°2	58°37	58°10	57°43
37	62°48	62°28	62° 8	61°48	61°27	61° 5	60 44	28 59 60 21	59°5	59°25	59°10	58 46	58 20	57°54	57°28	57° 1
-	65°11	61°51	61°30	61°9	60°48	60,26	60°4	29°39 59°41	59°18	58°54	58°30	58°5′	57°39	57°13	56 46	56° 19
38	27°49	28°9'	28 30	28 51	29 12	29 34	29 56	30°19 59°2	30 42	316	31 30	31°55	32°21	32 47	33°14	
39	28 27	2847	29 7	29°20	29 50	30 12	30°35	30 58	31°21	31 46	32 10	32°36	33° 2′	33 28	33 54	34 23
40	60 57 29 3	6056 29°4	60 15 2946	59 53 30 7	30°20	30°50	31°13	58° 24 31° 36	32°0′	32 25	32 50	33 16	33 41	34° 8	34 35	35°3′
41	60°26	60° 0° 30° 0	59°39 30°21	59°17 30°43	58°55 31°5	58°32 31°28	58°9 31°51	57 45 32 15	57 ๊2i 32 ๊39	56°57 33°3	56°32 33°26	56°6 33°54	55°39 34°21	55°,2 34°48	54°44 35°16	54°16 35°44
42	59°4¢	59°24	593	58 40	56 18	57°55	57°32	57°8′ 32°52	56°43	56 19	55°53	55°27	55°0'	54°33	54°5	53°37
	3015		3737	J. 20		523			/	''	•••	3		/	, - 54	

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TABLE 2.—(Continued.)

Angle of Edge. GEAR.

			,				GEA								
	56	55	54	53	<u>52</u>			_					44	43	42
12		77°42 12°18		77°15 12°45			76°30 13°30						74°45 15° 15	74°25	
13	76°56	76°42	76°28	76°13	75°56	75°40	75°26	75°8′	74°51	74°32	74°13	73°53	73°32	73°11 16°49	72°48
14	75°58	75°43	75°28	75°12	74°56	74°39	74°21	74°3′	73°44	73°25	73°4	72 43	72°21	71°58	71°34
	14° 2′	14° 17 74°44									16 56 71°56			18°2 70°46	
15	15° 0′	15 16	15°31	15 48	16°5	16 23	16 42	17°1′	17°21	17 42	18°4	18,56	18 50	19 14	19°39
16	74° 3	73°47 16°13	73 30 16 36	73 12 16 48	17°6′	17°25	17 45	18° 5′	18°26	18 46	19°11	19°34	19 59	69°35 20°25	20 51
17		72°49		72°13 17°47										68°26 21°34	
18	72°11	71°53	71°34	71°15	70°54	70°33	70°12	69°50	69°26	69° 3	68 [°] 38	68°12	67°45	67°17	66°48
	71°15	70°57	70°37	70°17	69°56	69°34	69°12	68°48	68°25	67°59	67°34	67°6	66°38	22 43 66°10	65°39
19		19° 1												23°50	
20	19°39	19°59	20°19	20°41	21° 3′	21°25	21°48	22 12	22°37	23° 3′	23°30	23 58	24° 27	24 57	25 28
21		69°0 20°54												63°58 26° 2	
22	68 33	68 12	67°50	67°27	67°4	66°40	66°15	65°49	65°23	64° 55	64°26	63°57	63°26	62°54 27°6	62°21
23	67°41	67°18	66°55	66°32	66 8	65°44	65°18	64°51	64°24	63°55	63°26	62,29	62°24	61°52	61018
<u> </u>														28 8 60°50	
24	23°12	23 34	23 58	24°22	24 46	25°12	25 38	26 6	26°34	27° 3	27°33	28 4	28 37	29 10	2945
<u> 25</u>	24° 3′	24° 27	24°51	25°15	25°40	26° 7	26°34	27° 2	27°31	28 1	28°31	29°3	29 36	59°50	30°46
26														58°50 31° 10	
27	64°16	63°51	63°26	63° 0′	62°34	626	61°38	61°8	60°38	60° 7	59°35	59°2	58°29	57°53	5716
	63°26	63° ı'	62 36	62 9	61 42	61°14	60°45	€0°15	59°45	59°13	58 40	58° 7	57°32	32° 7 56° 56	56°19
28														33°4	
29	27 23	27 48	28°15	2841	29 9	29 37	30°7	30°37	31° 8	31°41	32° 14	32 48	33°23	34° 0	34 37
30	28 11	28 37	29 3	2931	29 59	30 28	30 58	31 28	32 o	32 23	33°7′	33 41	34 17	55°5′ 34°55	35 32
31	61°2	60 26	60°6	59°41	59°12	58°42	58°12	5741	57° 6′	56°36	56° 1	55°26	54°50	54° 12 35° 48	53°34
32	60'15	59 48	59°21	58°52	58°34	57°54	57°23	56°52	56°19	55 45	55°11	54°35	53°58	53°21	52°42
_		30 12 59°2	30 39 58 34	31°8 58°5	31°26 57°36	32 6 57 6	32 37 56°34	33°8	33 41 55°30	34°15 54°56	34 49 54°21	35°25 53°45	36°2 53°8	36°39 52°29	37 16 51°56
33	30 31	30°58	31 26	31°55	32°24	32°54	33°26	33 58	34°30	354	35 39	36°15	36 52	37°31	38 9
34	58°44 31°16	31 44	32°12	32°41	33°11	33°4i	34°13	34°45	35°19	34 7 35 53	36°28	52 52 37 8	37 42	38 20	39 0
35	58° o	57°32 32°28	57° 3′ 32° 57	56°33 33°27	56° 3' 33° 57	55 32 34 28	55°0 35°0	54°26 35°32	53°54 36°6	53° 20 36° 40	52°44 37°16	52°e 37°≈	51°30	50°51 39° 9	50°12 39°48
36	57 16 32 44 56°20		_	_			_		_	_	_			_	
17) /	30 36	100.4	00,33	00,3	3.3		3330		- L- L-3	31.77	3	30 33	10 30	73.17	70 37
3/	33°28 55°51 34°9	33 56	34°25	34 55 54 22	35°26	35 58	3630	374	37 37	38°13	38 48	39°25	40°4	40 43	41°23
39	345	35°21	35 50	3621	36°53′	37 24	37 57	38°31	39 6	39°41	40°18	40 55	41°33	47°48 42°12	42 53
40														47°5 42°55	
41	53 48	53°17	52°48	52°16	51°45	SI°IE	50 39	50°5	49°30	48°54	46°17	47°40	47°1	46 22	45°41
40	36 12 53°8′	3643 52°38	37 12 52° 8'	37 44 51°36	38 15 51°4	38 48 50° 12	39°21	39 55 49°24	46 46	41 6	41 43 47°16	42°20	42°59	43 38	44 19
42	36°52	37°22	37°52	38 24	38 [°] 56	39°28	40°2	40 36	41°11	41°47	42°24	43° i	43 40	44°20	45°

TABLE 3.

ANGLE OF FACE. GEAR.

28 32° 32° 33° 33° 18 33° 73° 34° 33° 18 33° 73° 34° 33° 18 33° 1									AR.							
13		41	40	39	38	37	36	35	34	33	32	31	30	29	28	2
13	_	13°37	13.57	14 18	14 39	15 1	15 24	15 49	15 15	16 43	17 18	1743	18°15	18°51	18 27	20°
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T	1 4	16°13	16°54	16°59	17°24	17°50	18°17	18°45	19°16	19 48	20°20	20°56	21°34	22°16	22°55	23°3
1		1790	68°0	6729	18 44	19011	6547	65°s	64°30	6348	63'6	22,20	61°32	23.51	59°47	58
16 66 66 65 65 65 65 65	15	67°6	66°45	66914	6540	65°з	64°26	6349	63°8	62°24	61°39	60°61	60°2	59°a	58%	57
T	A			1					ı			1				
1		19 66	20°24	2031	21 021	2188	2214	22 67	23 33	24 10	250	2531	26°14	2759	2747	28
10 63% 63% 63% 62% 61% 61% 61% 61% 60% 59% 59% 59% 58% 57% 50% 55% 55% 54% 52% 54% 53% 54% 55%	1/	64 54	64 20	6346	63°s	62°31	6180	6109	6025	5940	5802	58°i	57°10	56°13	55°15	54°
19 22 3 2 4 2 2 4 2 2 3 2 2 4 2 2 3 2 2 4 1 2 2 5 1 2 5 3 7 2 6 1 3 2 6 2 2 3 2 2 3 2 2 3 3 3 4 3 3 4 3 2 2 3 3 3 4 3 3 4 3 2 2 3 3 2 2 4 1 2 4 3 2 5 6 2 5 4 2 6 6 6 2 6 2 6 2 6 2 6 2 6 2 6 2 6 2	18	21.9	2187	22°6	2238	23°s	23 43	24 18	24°66	2534	2615	26°57	2742	2829	29 18	30°
19 6236 62°1 61°2 60°4 60°4 59°1 59°1 59°3 57°4 56°4 55°1 54°2 53°8 52°8 50°6 60°3 60°3 60°4 59°4 50°6 25°6 25°6 25°6 25°6 25°6 25°6 25°6 25		22 20	22 49	2320	2352	24 26	25°1	2537	26°15	26 56	2730	28,55	29°s	29°56	3043	314
20 60°30 60°30 60°30 60°30 60°30 60°30 60°30 58°30 58°30 58°30 57°30 58°	13	6236	62°1	61024	60°44	60°4	5921	5837	57°51	57°4	5614	5522	54°26	53°28	52°28	512
2 24 3 9 25 % 25 % 26 % 27 3 0 28 7 0 28 8 0 29 8 1 30 7 1 31 7 4 31 8 2 22 8 1 33 8 4 2 25 % 25 % 25 % 25 % 25 % 27 8 6 28 8 1 27 8 0 28 8 1 28 2 30 6 30 8 1 30	20	23'30	241	24 32	256	2540	2616	26 55	2784	28 15	28 58	2944 54°a	30 31	5200	32 13	331
22 25% 26% 26% 27% 28% 28% 29% 29% 30% 31% 31% 32% 33% 34% 34% 34% 34% 34% 35% 3	21	24°39	25°10	2543	26°18	26°53	2730	2810	28 50	29°82	30 17	31°4	3182	3243	3386	34°2
26	-	60°25	59°46	59°7	58°26	5743	57°0	56°14	55°24	54°36	5348	52°50	51°52	50°53	49°60	484
23 26°52 27°26 28°0 28°0 28°0 28°0 28°0 28°0 28°0 28°0																
26 57; 5635 5635 563 550; 1542 5534 522 130; 3237 3433 350; 3652 3737 3433 350; 3650 3652 3737 3433 350; 3650 3652 3737 3433 350; 3650 3652 3737 3433 350; 3650 3652 3737 3433 350; 3650 3652 3737 3433 3633 3633 3633 3633 3633 3633																
26	LO	58 16	5738	56 56	56 14	5530	5445	53 67	53°B	52 16	51 24	50°28	49°29	48°30	4727	46°2
25	24	57.5	5636	5563	55°	54°1	53°	52°61	52°2	51%	50°s	493	ر ده	473.	440.0	450
26	25	28'59	2934	3002	30%9	31°29	32°10	3282	33°37	34°23	35°11	36°6	36°52	3747	3843	392
27 31°3 31°3 32°8 32°8 32°57 33°3 34°6 35°3 35°4 36°3 37°5 38°6 39°6 40°4 41°1 54°5 32°5 32°5 33°6 33°6 33°6 33°6 33°6 33°6 33°6 33	20	56 15	3038	31014	31 54	53 23 32 34	52 36 33°16	33 58	3446	50°5	49 n	37 10	3802	38 66	39 53	40°4
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29 32 **s 33 **s 34 **n 34 **s 35 **s 36 **s 37 **s 37 **s 38 **z 39 **s 240 **s 41 **n 42 **s 35 **s 36 **s 37 **s 38 **r 38 **r 38 **s 36 **s 36 **s 36 **s 37 **s 38 **r 38 **r 38 **r 38 **r 38 **s 36 **s		32°2	32 39	33°18	3357	3439	3521	36°7	36 62	3740	38°20	39%	40°15	4109	435	-
30 5227 5124 511 506 4920 4841 4730 4630 4664 450 4412 4371 4210 30 5133 5030 5057 4920 4830 4745 4635 4663 4563 4563 4614 4311 421 4210 31 3453 3531 3611 3611 3652 3735 3820 3955 3952 4031 4122 4223 32 3536 3627 376 3748 3831 3935 4001 4049 4138 4222 33 3639 379 3890 3842 3936 4652 4611 4322 4233 3639 379 3890 3842 3936 4652 4610 4049 4138 4228 33 3639 379 3890 3842 3936 4064 4558 458 4414 4233 34 3732 3811 3833 3935 4018 4124 4123 35 3639 379 3890 3842 3936 4084 4124 4123 36 3931 3962 4034 4115 426 458 458 4420 4339 4231 36 3931 3962 4034 4115 426 428 4381 4241 4384 424 4384 4253 37 4637 4524 463 4536 4421 4334 424 4334 424 4334 4255 458 4458 458 458 458 458 458 458 458		5322	52 39	5156	51°11	50,52	49 37	48 4.7	4756	47°2	46°6	4511	44"11	459	427	
30 3367 3436 357 3586 3688 3721 3867 3683 3943 4082 4123 4216 31 3483 3581 3681 3682 3735 3886 3885 3985 3983 4081 4182 4223 32 3584 3627 376 3748 3881 3955 4081 4484 4232 4223 33 3584 3627 376 3748 3881 3955 4081 4484 4232 4223 33 3689 379 3880 3884 3926 4080 4086 4184 4233 3689 3782 3884 3884 3926 4080 4084 4184 4233 3689 3782 3884 3884 3926 4080 4084 4184 4233 3689 3782 3884 3884 3926 4080 4086 4184 4233 3884 3926 4080 4086 4184 4233 3884 3926 4080 4086 4184 4233 3884 3884 3926 4080 4086 4184 4233 3884 3884 3926 4080 4086 4184 4233 3884 3884 3926 4080 4086 4184 4233 3884 3884 3926 4080 4086 4184 4233 4233 3884 3884 3926 4080 4086 4184 4189 4233 4233 4231 4231 4284 4888 4884 4884 4884 4884 4884 488														4213		
31 34°s3 35°s1 36°s1 36°s2 37°s5 38°s0 39°s 39°s2 40°4 41°s2 42°s3 32 35°s4 36°27 37°6 37°s 38°s1 39°s 40°s1 40°s9 41°s6 42°s9 49°s9 37°s 39°s 39°s 40°s1 40°s9 41°s6 42°s9 42°s9 48°s1 47°s8 46°s8 45°s9 45°s 44°s6 44°s8 42°s8 45°s9 46°s1 44°s9 41°s8 42°s8 48°s9 37°s 38°s1 38°s3 39°s5 40°0 40°s 41°s4 42°s9 42°s8 48°s9 48°s1 47°s2 46°s6 45°s8 45°s 44°s6 43°s8 45°s 44°s6 43°s8 45°s 44°s6 43°s8 45°s 44°s8 43°s6 42°s8 45°s 44°s6 43°s8 45°s 44°s6 43°s6 43°	20	33°57	3436	35°s	3586	36°38	3721	38°7	3683	3945	4032	4125	123		'	
31 50 ⁴ ₄₁ 49 ⁵ ₃₇ 49 ⁵ ₁₃ 48 ⁵ ₂₈ 47 ⁵ ₂₉ 46 ⁵ ₂₂ 46 ⁶ ₁₁ 43 ⁵ ₁₀ 44 ⁵ ₁₅ 43 ⁵ ₂₀ 44 ⁵ ₂₈ 49 ⁵ ₂₀ 49 ⁶ ₂₂ 47 ⁵ ₂₃ 46 ⁶ ₂₄ 45 ⁵ ₂₉ 46 ⁶ ₂₄ 46 ⁵ ₂₄ 4																
34 49 50 49 7 48 21 47 34 46 58 45 8 45 8 44 14 42 33 33 36 38 36 36 38 36 36 38 36 36 36 36 36 36 36 36 36 36 36 36 36	31	50°41	49°57	49°13	48°28	4739	4652	46°1	45°10	4415	4320	4225				
34 48°19 48°17 47°32 46°46 45°58 45°8 44°18 43°16 42°18 42°18 43°16 42°18 42°1	32	3546	3627	37%	3748	3831	39 15	40°1	4049	4138	4228					
34 48°12 47°27 46643 45°37 45°8 44°18 43°4 41°49 42°37 35 38°21 39°0 39°440°26 41°10 41°35 42°41 46°39 42°37 36 46°37 45°32 45°8 44°21 43°34 37 40°0 40°40 41°22 42°5 44°31 42°35 44°31 42°35 44°31 42°35 45°37 45°32 45°3 44°32 42°35 42°41 38 40°47 41°22 42°44 43°37 38 40°47 41°22 42°5 43°37 40°40 41°42 42°5 43°39 42°32 41°12 42°44 43°45 42°5 44°31 42°5 44°31 42°5 44°31 42°5 44°31 42°5 42°5 44°41 42°	22	36° 89	37%9	38%	38°42	3926	40°0	40%	41 44	4324						
34 4812 4727 46623 4561 458 4420 4329 4237 35 3882 3983 39844086 4180 4185 4724 4633 4564 458 4420 4381 4241 36 3981 3982 4034 4181 428 37 4080 4080 4182 428 424 4387 4582 458 4422 4387 4248 38 457 4424 4380 428 4286 4087 4424 4380 428	33	48°39	48°17	4732	46°46	45°58	45°B	44°18	43°26	4233						
35 38 22 39 5 39 34 40 26 41 10 41 33 42 41 35 42 42 41 39 51 40 5	34	4802	4727	46%3	4567	40 18	414	4330	4237							
36 39°11 39°62 40°34 41°15 42°0 46°37 45°32 45°6 44°21 43°34 37 40°0 40°40 41°22 42°5 45°32 45°6 44°22 43°37 38 45°7 44°24 43°39 44°24 43°40 42°4 42°54 43°6 42°54 43°6	25	38,55	39°3	3944	40026	41°10	4155	42°41		,						
36 46°37 45°52 45°6 44°21 43°34 42°45 37 40°0 40°40 41°12 42°5 45°52 45°9 44°22 43°57 42°52 45°9 44°22 43°57 42°52 45°9 44°22 43°57 42°52 45°9 44°22 43°57 42°52 45°9 44°22 43°57 42°52 45°9 44°22 43°57 42°52 45°6 44°24 43°69 45°7 44°24 43°69 42°56 46°43°16 45°16 45°16	30	4724	4633	45 34	45°8	44°20										
37 45\$\frac{5}{2}\$ 45\$\frac{6}{3}\$ 44\$\frac{2}{2}\$ 43\$\frac{3}{3}\$ 7 42\$\frac{4}{3}\$\frac{4}{3}\$ \frac{4}{3}\$\frac{2}{3}\$ \\ 38 \frac{4}{3}\$\frac{5}{7}\$ 44\$\frac{2}{4}\$ 43\$\frac{3}{3}\$ \\ 39 \frac{4}{1}\$\frac{5}{2}\$ 42\$\frac{5}{14}\$ 42\$\frac{5}{3}\$ \\ 40 \frac{4}{3}\$\frac{3}{16}\$ 1.50	36	46°37	1	1	4421		4245	٠.								
38 40 47 41 28 42 59 42 52 39 42 52 52 39 42 52 52 39 42 52 52 39 42 52 52 39 42 52 52 52 39 42 52 52 52 52 52 52 52 52 52 52 52 52 52	37	1		4122	4-2°5	4248										
38 45°7 44°24 43°39 42°52 39 44°24 43°49 42°56 44°24 43°49 42°56 44°24 43°49 42°56	20	40 47	4128													
39 44°24 43°40 42°56	38	45°7			4252											
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TU ₄₃₄₂ T4 58	10		0	-	}											
	ŤU	4342	74 58	j												
41 43°2	41	43°2														

TABLE 3.—(Continued.)

ANGLE OF FACE.

GEAR.

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	26	25	24	23	22	21	20	19	18	17	16	15	14	13	12
12	1	213	į.	ı		,		1 .	28 25			3244	ł	3016	13877
<u> </u>			24°15						3042			4524			
13)		5721		ľ				(•	45°21	43°20	41°9	38°₄a	1
14			26°8	27°5	284	29 9	30°20	3: 33	32°52	1	l .	37°28	130		
•••			55°38					48 47 33°36		4512		41°24	_	j	
15			53°58						4520		١.	139 3a			
16	2782	284	29°43	30°44	31°50	32°58	3412	3531	3684	38°23	39 57		•		
10	54 30	53°31	52°21	51°6	4946				43°38			j			
17	53°A	52°0	3126 5048	36 28	48011	46°47	36°0	3721 4303		40°15					
18			33°4						4031		,				
_			49°18						40 31	1					
19			34°38 47°54			•	39°24								
20			36°8			39 39	۰		1					•	-
4 U		-	46°32				4067								
21		36 32	3737 15°13		39°54		1								
22			39°0	40°8)								
22	46°24	45°13	43°58	4240	41 19										
23	1		40°20	11100											
24	39°29	40°32	4246		ļ										
4	44°3	42°52	4138												
25	40°43 42°57			•											
	41°53		•												

$$g_a = 90^{\circ} - (\alpha_a + \beta)$$

$$g_b = 90^{\circ} - (\alpha_b + \beta)$$

(See page 13.)

TABLE 4 ANGLE OF FACE.—GEAR.

	72	71			68			ì								57
12	7.°53 78°53				8° 21 78° 19											
13	8°40	8°48	8° 54	9, 5		9° 18′	9° 26	9° 35	9.43	9°52	10, 1,	10, 11	10° 21	10°31	10°42	10°53
IA	9°.26	9°34	9°42	9° 50	9'59	10 8	10°16	10.5	10:35	10°45	10°54	11° 5′	11 16	II° 27	11°39	11° 50
IE	10, 15,	19°21	10°35	10°39	76 43 10°47	10° 57	11° 6'	114 16	11° 27	Il° 37	11°47	11° 59	15, 11	12"22	12'35	74°14 12°48
10					75°55 11°37											73 18 13'45
16			75 3i	75 20	75 7 12°24	74 54	74 40	74 27	74°13	73 59	73°44	73° 28	73°13′		72°40 14°26	
W	75 io	74 58	74 46	74°33	74°20 13°12	74° 5′	73°52	73 38	73°23	73° 9	72 52	72° 35	72°21	72° 3′	71°45	
18	74 25		74"	73 46	73 32	73°19′	73° 4′	72 49	72 33	72 18	72°2′	71°44	71 28	71°10	70°51	70 34
19	73°40	73 27		73°	72 4 É	72°31		7 2°	71°45	71 28	71°11	70 54	70 30	70 17	69°59	69°39
20					14°46 72°											
21	14°43	14°55 71°59	15° 8′ 71°44	15°21 71°29	15°33′ 71°13′	15°46 70 58	15° 59 70° 41	16°13′ 70°25	16° 28' 70° 8'	16°42 69°50	16°58 69°32	17°13 69°13	17°28 68°54	17°46 68°34	18° 2' 68° 14	18°30 67°52
22	15° 27	15.40	5°53	16° 6'	16° 20 70° 28	15° 33	16° 47	17° 2'	17°16′	17°31	17°49	18, 3,	18° 20	18°37	18° 56	19'13
23	16, 15	16°24	16°38	16°51'	17° 5 69 43	17'26	17°34	17°50	18° 5'	18°26	18°36	18*54	19° 10	19°28	19°48	20'5
24	15°55	17° 9	17"22	17°37	17.51	18.6.	18°21	18° 37	18, 23	19° 9	19,56	19°44	20° 1	20,13	20,33	20,28
	17°39	17°52	18" 6"	18°21	68°59 18°36	18"52	19°7'	19°24	19'40	19 57	20014	20 32	20'51	21'10	21°29	21.50
25	10000				68°14 19°22											64 28 224i
28					67°30 20°6											
21	67 57	67°39	67 2Z	67 5	66°46	85 99	66° 8′	65 48	65°29	65° 8′	64 46	64 24	64° 1	63°39	63° 14	62.49
28	67 16	66 59	66 41	66 22	66°4'	65°44	65° 25'	65 5	6444	64°22	64° 1'	63 38	53 15	62°5ź	62 27	62 1
29	66° 35	66 17	65 59	65 40	63°21	55° 2	6442	64°2i	63°59	63 37	63°15	62 52	62 28	62'5	61'40	61°14
30	65 55	65 37	65 18	64 58	22°15 64°39	64 18	63 58	63°38	63 18	62 54	62°30	62" 7	61°43	61 18	60°54	60°27
31	21°50	22°6′	22 24	22°41	22°59' 63°57	23°17 63°37	23°35 63°15	23°55 62°55	24°14′ 62°32′	24°34 62°10	24°54 61°46	25°17 61°23	25°38′ 60°58′	25°58' 60°34	26°22′ 60°8′	25 46 53 42
32	25,31	22°48	23'4'	23°23	23°40' 63°.16'	23° 59'	24°18	24 38	24 58	25°18'	25°39	26°1'	26°23′	26°45	27.9	2734
33	53,10	23°28	23°46	24°4'	24° 22	24°41	25'1'	25°21	25°42	26° 2'	25°24	26°45	27°9'	27°31	27°56	28 19
3/		24° 8	24.27	24 44	25°4	25°23	25°42	26° 3'	26°24	26,46	27°7′	27'29	27°52	28°16	58,40	29°5
35	24 29	24°48	25'6'	25 25	61° 56	25 4	26°24	26°45	27°6′	27°28	27°50	28°13	28°36	1.62	29 25	29'50
	52, 9	25°27	25°45	26°5	61°16'	25°45	27° 5	27°26	27°43	28.10	28°33	28°56	29°26	29 43	30°9'	30'35
36		61°41	61°20	60°59	60 36	60°15	59°51	5928 28°7	59°4	58°40	58 15 29°15	57°50 29°30	57 24 30° 2	\$6°57	56°29 30°52	56°1′ 31°18
37	61 23	612	60 41	60 20	59 58 2744	59 35	59 13	58 49	58° 25	58 1	57 35	57 10	56 42	56 15	55 48	55 so
38	60 47	60 26	60 4	5942	59°20	58 56	58 34	58° 9′	5745	57ºzí	56°55'	56 30	56 2	55 35	55 7'	54 39
35	60°9	59 48	59 28	59 4	28°22 58°42	58 19	5755	51 31	57 7	56°41	56° 16	55 49	55°22	54 54	54 28	5357
40	59 34	59 38	58 50	58°27	29° 1′ 58° 5′	5742	57 17	56 54	56 28	56 2	55 37	55 10	54 44	54 15	53 46	53 18
41	58 57	58 37	58 15	57° 52	29°39′ 57°29′	57 4	56 40	56 15	\$5 51	55 25°	54° 59	54 32	54 4	53 36	53 7	52 38
42	53,25	29 12	29"33	29.22	30°16 56°52	30°38	31°	31'23	31.47	32° 10'	32°35	33°	33° 26	33, 25	34 19	34 46

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	56	55	54	53	52	51	50	49	48	47	46	45	44	43	42
12	10° 6'	10° 16	85 0	10° 39'	10°52	II° 3′	[[° 15	ll° 30	II°43	II° 58′	12° 13' 72° 59	12, Sa	12°45	13° 1′	13°19 71°25
13	11°4'	11° 16	11,58	11°42	11° 54	15°, 8.	12°20	12°37	12, 21	13° 7	13°23 71°49	13°40	13°58	14°16	14° 35
14	15° 5'	12,16	15,58	12°43	12.21	13 11	13°26	13°42	13°59	14° 15	14° 33 70 4 í	14°51	15° 10'	15°30	15° 51′
15	13° 1'	13°16	13° 26	13.43	13°59	14'14	14°36	14°47	15° 5′	15° 23	15°42	16° 1'	16° 22	16 43	17° 5′
16	13°59	14°13	14°28	14°44	15°1'	15°17	15° 35	15° 52	15 11	16° 30	69°34 16° 50	17°10	17°32	17°56	18° 18'
17	14°57	15"11	15" 23	15 44	16° 1'	16, 18,	16° 37	16°55	17°15	17°36	68° 28 17° 57	18,50	18°43	19° 6'	19°31
18	15°52	16°7'	16.56	16°42	17° 1'	17°20	17°39	17°58	18° 20	18°41	67°23' 19°3'	19°27	19° 50	20°18	20°42
19	16 49	17°2	17°23	17°41	18°	18°21	18°40	19° 1'	19°22	19°46	50° 8′ 66° 19	20°34	20 59	21°24	21 52
	17°44	18°1	18, 13	18°40	19°	19°20	19 41	20°2	20°25	20°49	65°16 21°13	21°39	22°5	22°32	23°
20	18° 39	18°57	19, 16,	19°37	19*58	20 19	20 41	21° 3'	21°27	21°52	64°13 22°17	2243	53,10	23°38	24 8
21	61° 31′ 19° 32′	67°9′ 19°52	66 46 20°12	66°23 20°33	65 58 20 55	65 33 21°17	65° 7'	64° 39	64 II 22°17	63°42 22°53	63°13 23°19	62 41 23°46	62 8 24 15	61 34	61° 25°14
22	66° 38	66 16	65°52	65 ั่ 27	65° 3′	64° 37	64 10	5341	63 13	62 43	62° 11' 24° 21	61 40	61°7	60°32	59 56
23	65° 47	65° 23	6458	64 33	64° 8′	63 4 i	53 13	62 44	62°15	61 44	61° 13 25°21	60°4i	60° 6'	59° 31	58 54
24	64 55	64° 3i	64° 5′	63 46	63 14	62°46	62 19	61 48	61 18	60° 47	60°15	59 4i	59 6	58°31	57 53
25	54° 5	63 39	63° 14	62 48	62°2i	61°53	61°24	60 53	60° 22	59 50	59 18 27°19	5844	58 9	57 32	56 54
26	63°15	62°49	62 2š	61 56	85 18	60°59	6036	59 59	59 27	58 55	58°21	5747	57 11	56 34	55 55
27	62 25	61 58	Sľ 32	6เ ร	60 37	60°7′	59° 38	59° 5′	58 aá	58°	57 26	56 sí	56 15	55 38	54 59
28	61°36	61° 9'	60 45	60 14	5946	59 16	58 45	58 13	57 42	57 8	29° 12	\$5°57	55°20	54 42	54 8
29	60° 47	60° 21	59°52	59°25	58 56	58 26	57 54	57 22	56 49	56 15	30° 8′ 55°40	554	54° 27	53 48	53° 9'
30	60°	59° 33	59°6′	58°36	58 6	57 36	57 4	56° 32	55°58	5524	31° 2' 54 48	54° 12	53° 34	52 54	52 15
31	59° 14	58°46	58°15′	57°49	57 18	58 47	56 15	55 42	55 8	54 34	31°55 53°57	53 21	52°42	52° 3′	51 23
32											32°46 53°8′				
33	5743	57 14	56 45	56 15	55 49	5ร์ เร	54° 39	54° 5	33° 32	58 5	33°38 52°20	51 42	51° 3	50 22	49 41
34											34°26 51°32				
35	30.12	30 42	31° 10	31°38	327	32°36	33°7	133°38	34°10	34°4	50° 45	35°5	36°27	37°5	37 42
36	31°	31°27	31°55	32°23	32 53	33°23	33° 53	34° 2	34°57	35°31	36° 5'	36°41	37°16	37°5	38°32
37	31'45	32,15	32°40	33° 8	33°38	34° 9	34°40	35°1	35°43	36°10	35°51 49°15	37°2	38°4	38°4	39°20
38	32°27	32°56	33°24	33°52	34 22	34°54	35°24	35°57	36°29	37° 3	37°38 48 32	38°14	38*5	39°28	40°7
39	33°10	33°35	34° 7	34°36	35°7	35°37	36° 9	36°4	37°15	37°4	38°24	39°	39°3	401	40%
40	33°52	34°21	34*50	35°18	35°49	36"2	36°53	37°2	37°58	38°3	39° 8	39°44	40°2	4058	41°37
41	34 33	35° 3	35°31	36° 1	36°31	37° 3	37° 35	38° 7	38'4	1 39	6 39 51	40°2	141° 5	41%	42"22
12	35°14	35°43	36°12	36,42	37'13	49 2°	48 5: 38 17	384	7 47 4 9 39°2	47 4 3 39°5	46°25 8'40°34 4'45°44	454 41°9	41°4	44 20	43 4
	151 30	50 59	120,58	14954	49 20	48 41	48 13	47 3	47 1	1462	45 46	45 7	442	143°44	1.4

PINION.

NATURAL SINE.

Deg.	0'	10'	20'	30′	40′	50′	60′	
0	.00000	.00291	.00581	.00872	.01163	.01454	.01745	88
1	.01745	.02036	.02326	.02617	.02908	. 03199	. 03489	88
$\frac{1}{2}$.03489	.03780	.04071	.04361	.04652	.04943	.05233	8
3	.05233	.05524	.05814	.06104	.06395	.06685	.06975	8
4	.06975	.07265	.07555	.07845	.08135	.08425	.08715	
5	.08715	.09005	.09295	.07545	.09874	.10163	.10452	88
6	.03713 .10452	.10742	.11031	.09364 $.11320$.03874 $.11609$.11898	.12186	8
7		.10742	.12764					8
	.12186			.13052	.13341	.13629	.13917	8:
8	. 13917	.14205	.14493	.14780	.15068	.15356	.15643	8.
9	.15643	.15930	.16217	.16504	.16791	.17078	.17364	80
10	.17364	.17651	.17937	.18223	.18509	.18795	.19080	79
11	.19080	. 19366	.19651	.19936	.20221	.20506	.20791	78
12	.20791	.21075	. 21359	. 21644	.21927	.22211	.22495	7
13	.22495	.22778	. 23061	.23344	.23627	.23909	.24192	76
14	.24192	.24474	.24756	.25038	. 25319	.25600	.25881	7
15	. 25881	. 26162	. 26443	.26723	.27004	.27284	.27563	7
16	.27563	.27843	. 28122	. 28401	.28680	.28958	.29237	7
17	.29237	. 29515	.29793	.30070	.30347	.30624	.30901	79
18	.30901	.31178	.31454	.31730	.32006	.32281	.32556	7
19	.32556	. 32831	.33106	.33380	.33654	.33928	. 34202	70
20	.34202	.34475	.34748	.35020	. 35293	. 35565	.35836	69
21	.35836	36108	.36379	.36650	.36920	.37190	.37460	
22	.37460	.37730	.37999	.38268	.38536	.38805	.39073	68
23	.39073	.39340	.39607	.39874	.40141	.40407	.40673	
24	.40673		.41204					66
		.40989		. 41469	.41733	.41998	. 42261	6
25	.42261	. 42525	. 42788	.43051	.43313	. 43575	.43837	64
26	.43837	.44098	.44359	. 44619	.44879	. 45139	.45399	6
27	.45399	.45658	.45916	.46174	. 46432	.46690	.46947	6:
28	.46947	.47203	.47460	.47715	.47971	.48226	.48481	6
29	.48481	.48735	.48989	.49242	.49495	.49747	.50000	60
30	. 50000	.50251	. 50503	. 50753	.51004	.51254	. 51503	59
31	. 51503	.51752	.52001	$.52\dot{2}49$.52497	.52745	.52991	58
32	.52991	.53238	.53484	.53730	.53975	.54219	.54463	5'
33	.54463	. 54707	.54950	.55193	.55436	. 55677	.55919	56
34	.55919	. 56160	.56400	.56640	. 56880	.57119	. 57357	53
35	.57357	. 57595	.57833	.58070	.58306	.58542	.58778	54
36	.58778	.59013	.59248	.59482	.59715	.59948	. 60181	5
37	.60181	.60413	.60645	.60876	.61106	.61336	.61566	5
38	.61566	.61795	.62023	.62251	.62478	.62705	.62932	5.
39	.62932	.63157	.63383	.63607	.63832	. 64055	.64278	5(
40	.64278	.64501	.64723	.64944	.65165	.65386	. 65605	49
41	.65605	.65825	.66043	.66262	.66479	.66696	. 66913	48
42	.66913	.67128	.67344		.67773			
				.67559		.67986	.68199	47
43	.68199	.68412	.68624	.68835	.69046	.69256	.69465	40
44	. 69465	.69674	.69883	. 7 0090	.70298	.70504	.70710	4
	60′	50′	40'	30'	20'	10'	0'	De

NATURAL COSINE.

NATURAL SINE.

0' .70710 .71934 .73135	10' .70916	20′	30′	40'	50′	60′	i .
. 71934							
		.71120	.71325	.71528	.71731	.71934	44
.73135	. 72135	. 72336	.72537	. 72737	. 72936	.73135	43
	. 73333	. 73530	.73727	. 73923	.74119	.74314	42
.74314	.74508	.74702	. 74895	.75088	.75279	.75471	41
.75471	. 75661	.75851	.76040	.76229	.76417	.76604	40
$.76$ $^{\circ}04$. 76791	. 76977	.77162	. 77347	.77531	.77714	39
.77714	.77897	. 78079	.78260	.78441	.78621	.78801	38
	.78979	. 79157					37
.79863	.80038		.80385	.80558	.80730	.80901	36
.80901	.81072		.81411	. 81580	.81748	.81915	35
.81915	.82081	.82247	.82412	.82577	.82740	.82903	34
.82903	.83066	.83227	.83388	.83548	.83708	.83867	33
.83867	.84025	.84182	.84339	.84495	.84650	.84804	32
.84804		.85111	.85264	.85415	.85566	.85716	31
.85716		.86014		.86310	.86456	. 86602	30
.86602	.86747	.86892	.87035	.87178	.87320	.87462	29
.87462	.87602	.87742	.87881	.88020		.88294	28
.88294	.88430	.88566	.88701	.88835	.88968	.89100	27
.89100	.89232	.89363	.89493	.89622	.89751	.89879	26
.89879	.90006		.90258	.90383	.90507	.90630	25
.90630	.90753			.91116		.91354	24
.91354	.91472		.91706	. 91821		.92050	23
.92050	.92163	.92276	. 92388			.92718	22
	.92827						21
.93358	.93461	.93565	. 93667	.93768	.93869	.93969	20
.93969	. 94068	. 94166	.94264	. 94360	.94456	.94551	19
.94551	. 94646	.94739			.95015	.95105	18
.95105	.95195				.95545		17
.95630	.95715	.95799			.96045		16
	.96205	. 96284			.96516		15
		.96741	.96814	.96887	.96958		14
	.97099						13
	.97502						12
							11
							10
							9
							8
$.99026^{+}$.99218	.99254	7
		.99323		.99389		.99452	6
	.99482	.99511		.99567	.99593	.99619	5
						.99756	4
		.99795					3
							2
							1
.99984	.99989	.99993	.99996	.99998	.99999	1.0000	0
60′	50′	40'	30′	20′	10'	0'	Deg.
	.75471 .76604 .77714 .78801 .79863 .80901 .81915 .82903 .83867 .84804 .85716 .86602 .87462 .88294 .89100 .91354 .92050 .92718 .93358 .9369 .94551 .95105 .95630 .96126 .96592 .97029 .97437 .97814 .98162 .98480 .98768 .99848 .99984	.75471 .75661 .76604 .76791 .77714 .77897 .78801 .78979 .79863 .80038 .80901 .81072 .81915 .82081 .82903 .83066 .83867 .84025 .84804 .84958 .85716 .85866 .86602 .86747 .87462 .87602 .88294 .88430 .89100 .89232 .89879 .90006 .90630 .90753 .91354 .91472 .92050 .92163 .92718 .92827 .93358 .93461 .93969 .94646 .95105 .95195 .95630 .95715 .96126 .96205 .96592 .96667 .97029 .97437 .97502 .97814 .97874 .98162 .98217 .98480 .98530	.75471 .75661 .75851 .76004 .76791 .76977 .77714 .77897 .78079 .78801 .78979 .79157 .79863 .80038 .80212 .80901 .81072 .81242 .81915 .82081 .82247 .82903 .83066 .83227 .83867 .84025 .84182 .84804 .84958 .85111 .85716 .85866 .86014 .86602 .86747 .86892 .87462 .87602 .87742 .88294 .88430 .88566 .89100 .89232 .89363 .89879 .90006 .90132 .90630 .90753 .90875 .91354 .91472 .91589 .92718 .92827 .92934 .93358 .93461 .93565 .93969 .9468 .94166 .94551 .94646 .94739 .95105 <t< td=""><td>.75471 .75661 .76977 .77162 .76004 .76791 .76977 .77162 .77714 .77897 .78079 .78260 .78801 .78979 .79157 .79335 .79863 .80038 .80212 .80385 .80901 .81072 .81242 .81411 .81915 .82081 .82247 .82412 .82903 .83066 .83227 .83388 .83867 .84025 .84182 .84339 .84804 .84958 .85111 .85264 .85716 .85866 .86014 .86162 .86602 .86747 .86892 .87035 .87462 .87602 .87742 .87881 .88294 .88430 .88566 .88701 .89100 .89232 .89363 .89493 .8979 .90006 .90132 .90258 .90630 .90753 .90875 .90996 .9153 .93276 .92388</td><td>.75471 .75661 .75851 .76040 .76229 .76604 .76791 .76977 .77162 .77347 .77714 .77897 .78079 .78260 .78441 .78801 .78979 .79157 .79335 .79512 .79863 .80038 .80212 .80385 .80558 .80901 .81072 .81242 .81411 .81580 .81915 .82081 .82247 .82412 .82577 .82903 .83066 .83227 .83388 .83548 .83867 .84025 .84182 .84339 .84495 .84804 .84958 .85111 .85264 .85415 .85716 .85866 .86014 .86162 .86310 .86602 .86747 .86892 .87035 .87178 .87462 .87602 .87742 .87881 .88020 .88294 .88430 .88566 .88701 .88835 .89100 .89232 .89363 .8949</td><td>.75471 .75661 .75851 .76040 .76229 .76417 .7604 .76791 .76977 .77162 .77347 .77531 .77714 .77897 .78079 .78260 .78441 .78621 .78801 .78979 .79157 .79335 .79512 .79688 .79863 .80038 .80212 .80385 .80558 .80730 .80901 .81072 .81242 .81411 .81580 .81748 .81915 .82081 .82247 .82412 .82577 .82740 .82903 .83066 .83227 .83388 .83548 .83708 .83867 .84025 .84182 .84339 .84495 .84650 .84804 .84958 .85111 .85264 .85415 .85566 .85716 .85866 .86014 .86162 .86310 .86456 .8602 .86747 .86892 .87035 .87178 .87320 .87462 .87602 .87742 <</td><td>.75471 .75661 .75851 .76040 .76229 .76417 .76604 .76604 .76791 .76977 .77102 .77347 .77531 .77714 .77714 .77897 .78079 .78260 .78441 .78631 .78801 .78861 .80938 .80212 .80385 .80558 .80730 .80901 .80901 .81072 .81242 .81411 .81580 .81748 .81915 .82903 .83066 .83227 .83888 .83548 .83708 .83867 .83867 .84025 .84182 .84339 .84495 .84650 .84804 .84958 .85111 .85264 .85415 .85566 .85716 .85662 .86747 .86892 .87035 .87178 .87320 .87462 .87462 .87602 .87742 .87881 .88020 .88157 .88294 .88294 .88430 .88566 .88701 .88835 .8968 .89100 .89273</td></t<>	.75471 .75661 .76977 .77162 .76004 .76791 .76977 .77162 .77714 .77897 .78079 .78260 .78801 .78979 .79157 .79335 .79863 .80038 .80212 .80385 .80901 .81072 .81242 .81411 .81915 .82081 .82247 .82412 .82903 .83066 .83227 .83388 .83867 .84025 .84182 .84339 .84804 .84958 .85111 .85264 .85716 .85866 .86014 .86162 .86602 .86747 .86892 .87035 .87462 .87602 .87742 .87881 .88294 .88430 .88566 .88701 .89100 .89232 .89363 .89493 .8979 .90006 .90132 .90258 .90630 .90753 .90875 .90996 .9153 .93276 .92388	.75471 .75661 .75851 .76040 .76229 .76604 .76791 .76977 .77162 .77347 .77714 .77897 .78079 .78260 .78441 .78801 .78979 .79157 .79335 .79512 .79863 .80038 .80212 .80385 .80558 .80901 .81072 .81242 .81411 .81580 .81915 .82081 .82247 .82412 .82577 .82903 .83066 .83227 .83388 .83548 .83867 .84025 .84182 .84339 .84495 .84804 .84958 .85111 .85264 .85415 .85716 .85866 .86014 .86162 .86310 .86602 .86747 .86892 .87035 .87178 .87462 .87602 .87742 .87881 .88020 .88294 .88430 .88566 .88701 .88835 .89100 .89232 .89363 .8949	.75471 .75661 .75851 .76040 .76229 .76417 .7604 .76791 .76977 .77162 .77347 .77531 .77714 .77897 .78079 .78260 .78441 .78621 .78801 .78979 .79157 .79335 .79512 .79688 .79863 .80038 .80212 .80385 .80558 .80730 .80901 .81072 .81242 .81411 .81580 .81748 .81915 .82081 .82247 .82412 .82577 .82740 .82903 .83066 .83227 .83388 .83548 .83708 .83867 .84025 .84182 .84339 .84495 .84650 .84804 .84958 .85111 .85264 .85415 .85566 .85716 .85866 .86014 .86162 .86310 .86456 .8602 .86747 .86892 .87035 .87178 .87320 .87462 .87602 .87742 <	.75471 .75661 .75851 .76040 .76229 .76417 .76604 .76604 .76791 .76977 .77102 .77347 .77531 .77714 .77714 .77897 .78079 .78260 .78441 .78631 .78801 .78861 .80938 .80212 .80385 .80558 .80730 .80901 .80901 .81072 .81242 .81411 .81580 .81748 .81915 .82903 .83066 .83227 .83888 .83548 .83708 .83867 .83867 .84025 .84182 .84339 .84495 .84650 .84804 .84958 .85111 .85264 .85415 .85566 .85716 .85662 .86747 .86892 .87035 .87178 .87320 .87462 .87462 .87602 .87742 .87881 .88020 .88157 .88294 .88294 .88430 .88566 .88701 .88835 .8968 .89100 .89273

NATURAL COSINE.

NATURAL TANGENT.

Deg.	0'	10'	20′	2 0′	40'	50′	60′	
0	.00000	.00290	.00581	.00872	.01163	.01454	.01745	89
$\overset{\circ}{1}$.01745	.02036	.02327	.02618	.02909	.03200	.03492	88
2	.03492	.03783	.04074	.02010	.04657	.04949	.05240	87
3	.05240	.05532	.05824	.06116	.06408	.06700	.06992	86
$\frac{3}{4}$.06992	.07285	.07577	.07870	.08162	.08455	.08748	85
5	.08748	.09042	.01311 $.09335$.09628	.09922			84
6	.10510	.10804	.09333	.11393	.11688	.10216	.10510	
7		.10504 $.12573$.13165	.13461	.11983	.12278	83
	.12278		.12869			.13757	.14054	82
8	.14054	.14350	.14647	.14945	.15242	.15540	.15838	81
9	.15838	.16126	.16435	.16734	.17033	.17332	.17632	80
10	.17632	.17932	.18233	.18533	.18834	.19136	.19438	79
11	.19438	. 19740	.20042	.20345	.20648	.20951	.21255	78
12	.21255	.21559	.21864	.22169	.22474	.22780	. 23086	77
13	.23086	. 23393	.23700	.24007	.24315	.24624	.24932	76
14	.24932	.25242	.25551	.25861	.26172	.26483	.26794	75
15	.26794	.27106	.27419	.27732	.28046	.28360	.28674	74
16	.28674	.28989	.29305	.29621	.29938	. 30255	.30573	73
17	.30573	. 30891	. 31210	.31529	.31850	.32170	. 32492	72
18	.32492	. 32813	.33136	. 33459	.33783	.34107	.34432	71
19	.34432	.34758	.35084	.35411	.35739	.36067	.36397	70
20	.36397	. 36726	.37057	.37388	.37720	.38053	.38386	69
21	.38386	.38720	.39055	. 39391	.39727	.40064	.40402	68
22	.40402	.40741	.41080	.41421	.41762	.42104	.42447	67
23	.42447	.42791	.43135	.43481	.43827	.44174	. 44522	66
24	.44522	.44871	.45221	.45572	.45924	.46277	.46630	65
25	46630	.46985	.47341	47697	.48055	.48413	.48773	64
26	.48773	.49133	.49495	49858	.50221	.50586	.50952	63
27	.50953	.51319	.51687	.52056	.52427	.52798	.53170	62
28	.53170	.53544	.53919	.54295	.54672	.55051	.55430	61
29	.55430	.55811	.56193	.56577	.56961	.57347	.57735	60
30	.57735	.58123	.58513	.58904	.59297	.59690	.60086	59
31	.60086	.60482	.60880	.61280	.61680	.62083	.62486	58
32	.62486	.62892	.63298	.63707	.64116	.64528	.64940	57
33	.64940	.65355	.65771	.66188	.66607	.67028	.67450	56
34	.67450	.67874	.68300	.68728	.69157	.69588	.70020	55
35	.70020	.70455	.08500 $.70891$.71329	09137 071769	.72210		
							.72654	54
36	.72654	.73099	.73546	.73996	.74447	.74900	.75355	53
37	.75355	. 75812	.76271	. 76732	.77195	.77661	.78128	52
38	.78128	. 78598	.79069	.79543	.80019	.80497	80978	51
39	.80978	.81461	.81946	.82433	.82923	.83415	.83910	50
40	.83910	.84406	.84906	.85408	.85912	.86419	.86928	49
41	.86928	.87440	.87955	.88472	.88992	.89515	.90040	48
42	. 90040	.90568	. 91099	.91633	.92169	.92709	.93251	47
43	.93251	. 93796	.94345	.94896	. 95450	.96008	.96568	46
44	.96568	.97132	. 97699	.98269	.98843	.99419	1.0000	45
	60′	50'	49'	30'	20'	10'	0'	Deg

NATURAL COTANGENT.

NATURAL TANGENT.

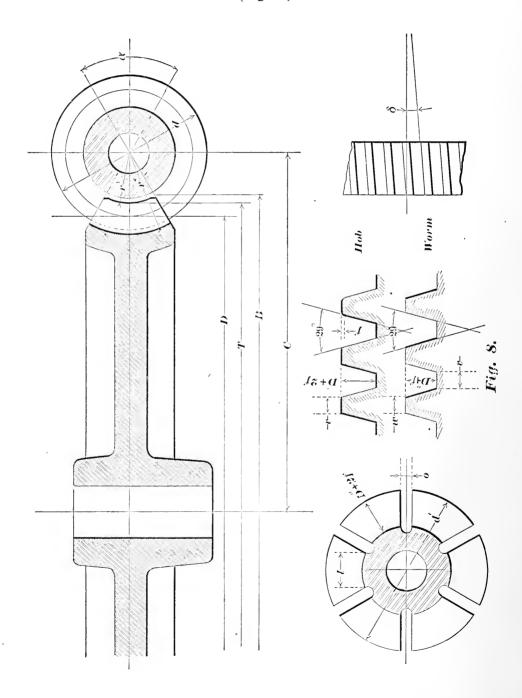
Deg.	0'	10'	20'	30'	40'	50′	60	
45	1.0000	1.0058	1.0117	1.0176	1.0235	1.0295	1.0355	44
46	1.0355	1.0415	1.0476	1.0537	1.0599	1.0661	1.0723	43
47	1.0723	1.0786	1.0849	1.0913	1.0977	1.1041	1.1106	42
48	1.1106	1.1171	1.1236	1.1302	1.1369	1.1436	1.1503	41
49	1.1503	1.1571	1.1639	1.1708	1.1777	1.1847	1.1917	40
50	1.1917	1.1988	1.2059	1.2131	1.2203	1.2275	1 2349	39
51	1.2349	1.2422	1.2496	1.2571	1.2647	1.2723	1.2799	38
52	1.2799	1.2876	1.2954	1.3032	1.3111	1.3190	1.3270	37
53	1.3270	1.3351	1.3432	1.3514	1.3596	1.3680	1.3763	36
54	1.3763	1.3848	1.3933	1.4019	1.4106	1.4193	1.4281	35
55	1.4281	1.4370	1.4459	1.4550	1.4641	1.4733	1.4825	34
56	1.4825	1.4919	1.5013	1.5108	1.5204	1.5301	1.5398	33
57	1.5398	1.5497	1.5596	1.5696	1.5798	1.5900	1.6003	32
58	1.6003	1.6107	1.6212	1.6318	1.6425	1.6533	1.6642	31
59	1.6642	1.6753	1.6864	1.6976	1.7090	1.7204	1.7320	30
60	1.7320	1.7437	1.7555	1.7674	1.7795	1.7917	1.8040	29
61	1.8040	1.8164	1.8290	1.8417	1.8546	1.8676	1.8807	28
62	1.8807	1.8940	1.9074	1.9209	1.9347	1.9485	1.9626	27
63	1.9626	1.9768	1.9911	2.0056	2.0203	2.0352	2.0503	26
64	2.0503	2.0655	2.0809	2.0965	2.1123	2.1283	2.1445	25
65	2.1445	2.1609	2.1774	2.1943	2.2113	2.2285	2.2460	24
66	2.2460	2.2637	2.2816	2.2998	2.3182	2.3369	2.3558	23
67	2.3558	2.3750	2.3944	2.4142	2.4342	2.4545	2.4750	22
68	2.4750	2.4959	2.5171	2.5386	2.5604	2.5826	2.6050	21
69	2.6050	2.6279	2.6510	2.6746	2.6985	2.7228	2.7474	20
70	2.7474	2.7725	2.7980	2.8239	2.8502	2.8770	2.9042	19
71	2.9042	2.9318	2.9600	2.9886	3.0178	3.0474	3.0776	18
72	3.0776	3.1084	3.1397	3.1715	3.2040	3.2371	3.2708	17
73	3.2708	3.3052	3.3402	3.3759	3.4123	3.4495	3.4874	16
74	3.4874	3.5260	3.5655	3.6058	3.6470	3.6890	3.7320	15
75	3.7320	3.7759	3.8208	3.8667	3.9136	3.9616	4.0107	14
76	4.0107	4.0610	4.1125	4.1653	4.2193	4.2747	4.3314	13
77	4.3314	4.3896	4.4494	4.5107	4.5736	4.6382	4.7046	12
78	4.7046	4.7728	4.8430	4.9151	4.9894	5.0658	5.1445	11
79	5.1445	5.2256	5.3092	5.3955	5.4845	5.5763	5.6712	10
80	5.6712	5.7693	5.8708	5.9757	6.0844	6.1970	6.3137	9
81	6.3137	6.4348	6.5605	6.6911	6.8269	6.9682	7.1153	8
82	7.1153	7.2687	7.4287	7.5957	7.7703	7.9530	8.1443	7
83	8.1443	8.3449	8.5555	8.7768	9.0098	9.2553	9.5143	6
84	9.5143	9.7881	10.078	10.385	10.711	11.059	11.430	5
85	11.430	11.826	12.250	12.706	13.196	13.726	14.300	4
86	14.300	14.924	15.604	16.349	17.169	18.075	19.081	3 2
87	19.081	20.205	21.470	22.904	24.541	26.431	28.636	2
88	28.636	31.241	34.367	38.188	42.964	49.103	57.290	1
89	57.290	68.750	85.939	114.58	171.88	343.77	œ	0
	60'	50	40'	30′	20'	10'	0,	Deg.
								!

NATURAL COTANGENT.

CHAPTER IV.

WORM AND WORM WHEEL.

(Fig. 8.)



FORMULAS.

L = lead of worm.

N = number of teeth in gear.

m =threads or turns per inch in worm.

d = diameter of worm.

d' = diameter of hob.

T = throat diameter.

B = blank diameter (to sharp corners).

C = distance between centers.

o = thickness of hob-slotting cutter.

l =width of lands at bottom.

b = pitch circumference of worm.

v =width of worm thread tool at end.

w =width of worm thread at top.

P = diametral pitch.

P' = circular pitch.

s = addendum and module.

t =thickness of tooth at pitch line.

 t^n = normal thickness of tooth.

f = clearance at bottom of tooth.

D'' =working depth of tooth.

D'' + f = whole depth of tooth.

 δ = angle of tooth of worm wheel with its axis, or the angle of thread of worm with a line at right angles to its axis.

If the lead is for single, double, triple, etc., thread, then

$$L = P'$$
, 2 P' , 3 P' , etc.

$$\alpha = 60^{\circ} \text{ to } 90^{\circ}$$

$$L = \frac{I}{m}$$

$$P' = \frac{\pi T}{N+2}$$

$$D = \frac{N P'}{\pi} = \frac{N}{P}$$

$$T = \frac{N}{P} + 2 s$$

$$b = \pi (d-2 s)$$

$$\tan \delta = \frac{L}{b} \quad \begin{cases} \text{Practical only when width of wheel on wheel pitch circle is not more than } \frac{1}{3} \text{ pitch diameter of worm.} \end{cases}$$

$$t^{n} = t \cos \delta$$

$$t^{1} = \frac{d}{2} - 2 s$$

$$t^{2} = t^{1} + D'' + f$$

$$t^{2} = \frac{D + d}{2} - s$$

$$t^{3} = \frac{1}{2} - s$$

$$t^{3} = \frac{1}{2} + \frac{$$

Note.—The notations and formulas referring to tooth parts, given on page 5 for spur gears, apply to worm wheels, and are here used.

Note.—Hob and worm should be marked, as per example:

4 turns per 1" single .25 P; .25 L.

2 turns per 1" double .25 I"; .50 L.

UNDERCUT IN WORM WHEELS.

In worm wheels of less than 30 teeth the thread of the worm (being 29°) interferes with the flank of the gear tooth. Such a wheel finished with a hob will have its teeth undercut. To avoid this interference two methods may be employed.

First Method. - Make throat diameter of wheel

$$T = \cos^2 14 \frac{1}{2}^{\circ} \frac{N}{P} + 4s$$
 or $T = \frac{.937 N}{P} + 4s$

This formula increases the throat diameter, and consequently the center distance. The amount of the increase can be found by comparing this value of T with the one as obtained by formula on page 36. To keep the original center distance, the outside diameter of the worm must be reduced by the same amount the throat diameter is increased.

Second Method.—Without changing any of the dimensions we found by the formulas given on page 36, we can avoid the interference to be found in worm wheels of less than 30 teeth by simply increasing the angle of worm thread. We find the value of this angle by the following formula:

Let there be

 $2 \gamma = \text{angle of worm thread.}$ N = number of teeth in worm wheel.

$$\cos \gamma = \sqrt{1 - \frac{2}{N}}$$

From this formula we obtain the following values:

As this latter formula involves the making of new hobs in many cases, on account of change of angle, we prefer to reduce the diameter of worm as indicated by first method, if the distance of centers must be absolute.

TABLE OF ANGLES FOR GASHING WORM WHIELS.—SINGLE THREADED.

.1000" .1111" .1250" .1333" .1429" .1538	- :1 t >	3-38 3-51 4-	3-2' 3-14' 3-	2-36, 2-47, 22	2017 2026 203	2-2' 2-10' 2-10'	1-49' 1-57' 2-	1230/1246/12	1-21/ 1-31/ 1-37/ 1-7	1215' 1224' 1200' 120	1213 1223 12	1-13 1-13 1-9	128, 1-13, 1-	124, 129, 12	151, 150, 15	58 151 15	64, 58, 12
29, 1538	9	4-10 4-29	3-28 3-44	2-58 3-12 3	2-56 2-48	2-30	225' 2215' 9	1-54, 5-5,	1-44' 1-52'	1-36 1-44	1-29 1-30	1223 1200	1-18' 1-24'	1-14' 1-19'	120, 1216,	120, 1211,	1-3' 1-7'
1066, 18	17	4-51 5-	4-3,4	3-20 3-3	3-2'-3	2-42 2-	2-26, 5-	2-13 2-	2-1, 2-	1-52 2	1244' 1264'	-1,22,1-1	1531 1539	1226 1234	1-21, 1-	1-17/ 1-3	1213/12
.1666". 1818". 2000". 2222". 2600". 2867". 3333". 3636". 3756". 4000". 4285". 4444". 5000". 5714". 10006". 6668". 7500". 8000"1.0000"1.33331. 5000"2.0000"3.0000"	10 10	5-18 5-49	4225' 4751' 5723' 674' 6755' 873'	0.20 3247 4-10 4-37 5-12 6-567 6-66 7-32 7-46 8-17 8-52	3-19/3-39/4-3/	-42' 2-57' 3-14' 3-30'	2-30' 2-56' 3-14' 3-30' 4-10' 4-51' 5-17' 5-27' 5-40' 0-14' 6-27' 7-15	2-26 2-30 2-67 3-10 3-10 3-47 4-26 4-40 4-50 5-17 5-40 6-50 6-36	2-1' 2-13' 2-20' 2-42' 3-2'	2-2' 2-15'		07' 1246' 1-57'	1-49,	34' 1-43' 1-54'	1-28′ 1-37′	1224' 1332' 1242'	1-13' 1-20' 1-27' 1-37' 1-49' 2-6' 2-26' 2-30' 2-44' 2-65' 3-7'
.2222" .25	4 1 4	0-28	5-23 6-	4-37 5-1	423 42	3-36 4-3	3-14/ 3-6	2-67′3-	2-42 3-5	2-15' 2-30' 2-48' 3-12' 3-44'	2-6 2-10 2-00 2-00 2-00 3-20 3-47 3-54 4-10 4-27 4-37 5-12 5-55 6-14	2-10' 2-20' 2-47' 3-14' 3-32' 3-30' 3-63' 4-10' 4-19' 4-51' 6-32'	2-2		1-48′ 2-2′	1242' 126	1-37' 1-4
0, 2857	- 21 33	7-16 8-17 0-38	4 6-55	2 6-50	4-33 6-12	3, 4-37,	30 4-10	9, 3-47	3-28 4-3	18,3-12,	30 2-58	26 2-47	2-17 2-36 3-2'	2-9' 2-27' 2-62' 3-7' 3-13' 3-26' 3-40' 3-46' 4-17' 4-64' 6-8'	2, 2-19,	1-65' 2-12' 2-33'	9, 2-6,
.3333	93	0-38, 10	8-3	6-55	6-3	6-93	4-51	4-25	423,	3-44	3-20	3-14,	3-2	2-52	2-19' 2-42' 5	2-33	2-26
3636	21 21	10-30 10-49 11-31	2.46 9	-32, 2-	30,00	3.52	-17 5-	-40,4-	-25 4-	4.	-247 3-	.32, 3-	-10 3-	2.7, 3	2-57 3-2	2-47 2-55- 3-4'	230, 25
750 .40	51 51	49,11-	-3 , ₉₋ 5	46 8=1	48 7-1	4, 6-2	27, 6-7	50, 5-1	33, 4-6	12 4-2	54 4-1	30, 3-6	25 3-3	13 3-2	2, 3-14,	65-3-	44′ 2-6
00, 1280	- 21 - 21 - 2	31,	8-46 9-3 9-38 0-18	7 8-52	6-36 6-48 7-15 7-47 8-3	4-37 6-23 5-52 6-4 6-27 6-55 7-10	19, 0-14	1 5-40	51, 6-12	9/4-48	0 4-27	53 4-10	30, 3-54	0,3540	4, 3,28		5, 3-7,
.4444					8-3	, 1 - 10,	6-27	, 29-9	4-26/4-33/4-61/6-12/6-23/6-3	4-4 4-12 4-29 4-48 4-50 5-36 6-23 6-42	4-37/	1-10	3-10 3-25 3-39 3-54 4-3 4-33 5-12	3-40,	328' 3-36' 4-3'	3-17' 3-26' 3-50' 4-23' 4-36'	3-14
5000,	91		i.				7-15	36,4	D-3 (G-	5-30 0-	5-12' 6	-51, 6	1-33 5	1217/ 45	4-3, 45	3-50, 4	3-39/ 4
714, .000	\$1 °C								6-65	23, 6-4	55 651	.32 6-4	12' 6-27'	.54, 6-8	37, 426	23 4-3	10 422
,,0666	- 01 										<u>, </u>	5-40 6-27	7, 6-3,	6-42 6-25	4-37 4-51 6-23 6-3	0-0	3-14 3-39 4-10 4-22 4-51 5-27 5-49
.7500".8	= :: ==													6-25	0-3	5-44 0	5-27' 5
3000,1.0																, 1-0	249
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TABLE OF ANGLES FOR GASHING WORM WHEELS.—SINGLE THREADED.

LEAD,	TURNS PER INCH.	6.1	C1	61	က	က	ော	ಐ	JME ₩	4	TCH	1d	10	7.0	1.0	70	ဗ
	7. IIO	8 42,	% 4	8 33,	33,	4 34'	32,	30,		1-14	61	60		114	- 01	∞ 4	
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11,"11		46, 5	44, 5	42' 4		38, 4	35,			(3)							
1000", 1111", 1250", 1333", 1429", 1538", 166	8 2	52, 20	50, 5	40, 51	46, 40	42, 45	30, 42	36, 36	34,	32, 27	22,						
33 .14	1 61	56, 1	53' 57	51, 5,	40, 52,	45, 48	42, 45,	39, 45,	22, 20,	24' 37'	2′ 25′	3.					
29".15	© ~	ا ه		54, 50		48' 52'	5, 48	2, 45		7, 40,), S	Ċ				
38″.16	F= 01	4, 10	101, 11,	53, 19	56, 12,			5, 43	42' 4(37' 40'		34, 20,	83			
36" ,1818"	e2 e2	9, 10	1-6' 1-12'	3, 1-9,	1-6	56 1-1	52, 57	, 23,	46' 50'	43' 47')' 44'	53' 42'	, 40,	25, 23,	30,	7	
18" .2000"	# PG	16/12-2	2′ 1°-20′	1-10	, 1-13,	, 1-7	, e-1	, 28,), 55,	,, 52	1, 40	, 40,	, 44,	3, 42,	,07 ,1	. 83	
0.2222"	#	23 1233)' 1-28'	, 1-25	1-21	1-15,	,6-1	, 1-5,	<u>T</u> .	, 22,	, 54,	2,	49,	40,	44	45,	40,
2, 2500"	1 2	3' 1-44'	8 1-39	5, 1-35,	1, 1=31,	5' 1-24'	1-18	1-13	1-8,	1-4′	<u>-1</u>	, 58,	, 55,	, 55,	50,	, 48,	46,
0".2857".	ත	4 1-59	9' 1º54'	5' 1-49'	1 12 44′	4 1-36	3' 1° 29'	3' 1-23'	1-18	1-14	,6-1	,9-1	1,3,	1-1,	57,	54′	52,
7, 3333	-101 CO	9, 2-19,	4' 2-13'	3' 2-7'	t' 2º2'	3' 1-52'	9' 1244'	3' 1-37'	, 1-31,	1-26	, 1-21	, 1-11	1-13,	1-9,	1-6,	1-3,	<u>,T</u>
3 .3636"	61	, 2-31	3' 2-25'	, 2-18′	2-13	2-2,	1-54	, 1-46	1-39	1-34	, 1-28	, 1-24	1-20	1-16	1-12	1-6,	,9-1
6 .3750″	01 01	2-36	, 2-29	, 2-23	3' 2=17'	2-6	1'-1-57'	, 1249′	1-43	1, 1-37	3, 1-31,	, 1-26	1-22′	1-18'.	, 1-15	11,	1-8,
0 .4000"	67 (C)	3' 2-47'	9, 2-39,	3′ 2° 32′	, 2-26'	2-14	,'.2-5'	, 1-57'.	1-49	,' 1-43'	1-37	, 1-32	1-28	1-23	, 1-20	1-16	1-13,
,,4285	61	, 2º59'	, 2-50	2-43	, 2-36	2-24	2-14	, 2-5	, 1-55	, 1250	1244	1-37	1-34	, 1-29	, 1-25	1-22′	, 1 - 18′
5 .4444"	6.1	325	2-57	2-49	2-42	, 2-30	2-19	2-10	2-2'	1-54	,1-48,	1-242	1-37	1-33	1-28	1-24	1-21
,,2000,,	E/4 61	3-28	3-19	3-10	3-2	, 2-48	, 2-36	, 2-26′	2-17	2-9,	, 2-2,	1-55	1-49	1-44	, 1-39	′;1 ² 35′	1-31
5".5714"	1 3	3-58	3-47	, 3-37	3~28	3 3-12	3 2-59	3' 2-47'	, 2-36	2-27	2-19	, 2-12	2-5	1-59	1-54	1-49	1-44′
,,'''		4-10	3-58	3-48	3-39	3-22	3-7'	, 2-55,	2-44	2-34	, 2-26′	2-18	2-11,	, 2-5,	1-59	1-54	, 1249′
, '9999',	- 01	4-37	4-25	4-13	423'	3-44	3~28′	3-14	3-2,	, 2-52,	2-42	2-33	2-26	2-19	2-13	2-7'	, 2-1,
,7500	100	5-12	4-58	4-45	4-33	4-12	3-54	3-39'	3-25	3-13	3-2	2-53	2-44	2-36	2-29	2-23	2-17
,8000,	™	5~32	5-17	5-4'	4-51	4-29′	4-10'	3-53	3-39	3~26	3-14	3-4	2-55	2-47	2-39	2-32	2-26
1.0000	₩	6-55	05≘36′	6-19	6-3	5-36	5-12	4-51′	4-33	4-17	4-3,	3-50	3-39	3-28	3-19,	3-10	3-2,
1.33331.	es 4			8-24	8-3,	7-26′	6-54	6-27	6-4	5-42	5-23	5-6	4-51	4-37	4-25	4-13	4-2'
1.5000	21 23				9-3,	3-22′	7-46	7-15	6-49	6-26	6-4,	6-44	5-27	5-12	4-58	4-45	4-33
.50002.00003.0000	⊷ ≎1						10-19	9-38	9-3,	8-32	8-3,	7-38	7-15	6-54	6-36	6-19	6-3
3.000									13-26	12-40	11-59	11-22	10-49	10-19	9-51	9-26	9-2

CHAPTER V.

SPIRAL OR SCREW GEARING.

(Figs. 9, 10, 11.)

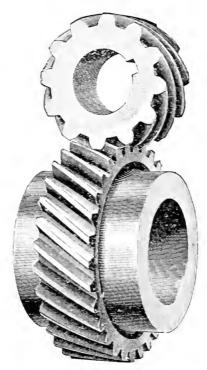


Fig. 9.

RIGHT HAND SPIRAL GEARS.

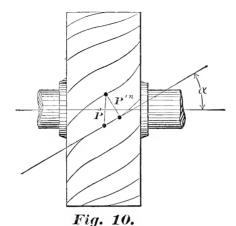
In spiral gearing the wheels have cylindrical pitch surfaces, but the teeth are not parallel to the axis. The line in which the pitch surface intersects the face of a tooth is part of a screw line, or helix, drawn at the pitch surface. A screw wheel may have one or any number of teeth. A one-toothed wheel corresponds to a one-threaded screw, a many-toothed wheel to a many-threaded screw. The axes may be placed at any angle.

Consider spiral gears with:

I. Axes parallel.

II. Axes at right angles.

III. Axes any angle.



LEFT HAND SPIRAL GEAR.

Let there be:

 $\begin{bmatrix}
N_a = \\
N_b =
\end{bmatrix}$ number of teeth in gears $\begin{cases}
a \\
b
\end{cases}$

C = center distance.

P = diametral pitch

P' = circular pitch.

 P^n = normal diametral pitch.

 $P'^n = normal circular pitch.$

 $\gamma =$ angle of axes.

 $L_1 =$ exact lead of spiral on pitch surface.

 L_2 = approximate lead of spiral on pitch surface.

T = number of teeth marked on cutter to be used when teeth are to be cut on milling machine.

D = pitch diameter.

B = blank diameter.

 $\alpha_a = \begin{cases} \alpha_a = \\ \alpha_b = \end{cases}$ angle of teeth with axis

t =thickness of tooth.

s = addendum and module.

D'' + f = whole depth of tooth.

Note.—Letters a and b occurring at bottom of notations refer to gears a and b.

I.—AXES PARALLEL.

Gears of this class are called twisted gears. The angle of teeth with axes in both gears must be equal and the spirals run in opposite directions. The angles are generally chosen small (seldom over 20°) to avoid excessive end thrust. End thrust may, however, be entirely avoided by combining two pairs of wheels with right and left-hand obliquity. Gears of this class are known as Herringbone gears. They are comparatively noiseless running at high speed.

II.—Axes at Right Angles.

Here we must always have:

- 1. The teeth of same hand spiral;
- 2. The normal pitches equal in both gears; and
- 3. The sum of the angles of teeth with axes = 90° .

CHOOSING ANGLE OF TEETH WITH AXES.

- I. If in a pair of gears the ratio of the number of teeth is equal to the direct ratio of the diameters, i. e., if the number of teeth in the two gears are to each other as their pitch diameters, then the angles of the spirals will be 45° and 45°; for, this condition being fulfilled, the circular pitches of the two gears must be alike, which is only possible with angles of 45°. In such a combination either gear may be the driver.
- 2. If the ratio of the diameters determined upon is larger or smaller than the ratio of the number of teeth, then the angles are:

 $\tan \alpha_a \equiv \frac{\mathrm{D}_a \; \mathrm{N}_b}{\mathrm{D}_b \; \mathrm{N}_a} \qquad \tan \alpha_b \equiv \frac{\mathrm{D}_b \; \mathrm{N}_a}{\mathrm{D}_a \; \mathrm{N}_b}$

In such gears the velocity ratio is measured by the number of teeth, and not by the diameters.

3. Given N_a , N_b and C:

If P_{a} is made = P_{b} , then we have case "1" and

$$\mathbf{P}' = \frac{\pi \ \mathbf{C}}{\frac{1}{2}(\mathbf{N}_a + \mathbf{N}_b)}$$

But if P_a' is assumed, then:

$$P_{b}' = \frac{C \pi - \frac{1}{2} N_{a} P_{a}'}{\frac{1}{2} N_{b}}$$

and

$$\tan \alpha_a = \frac{{\rm P}_a{}^{'}}{{\rm P}_b{}^{'}} \qquad \tan \alpha_b = \frac{{\rm P}_b{}^{'}}{{\rm P}_a{}^{'}}$$

The gear whose P' or α is larger will ordinarily be the driver, on account of the greater obliquity of the teeth.

4. Given N_a , N_b and C or D.

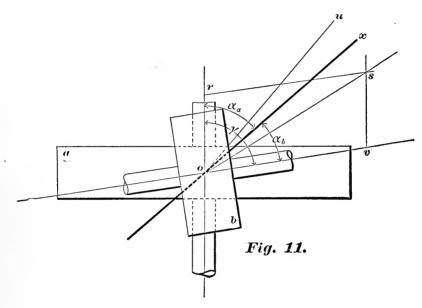
See case "7" under III., considering $y = 90^{\circ}$.

III.—Axis at any Angle (γ) .

- 5. Given case "1," under II., then angles of spirals = $\frac{1}{2}\gamma$, for the same reason.
- 6. Analogous cases to "2" and "3," under II., may be worked out, when angles of axes $= \gamma$, but they have been

omitted, partly because the formulas are too cumbersome, and partly because they are to some extent covered by cases "5" and "7."

7. Given N_a , N_b and C, or one of the pitch diameters. We find the angles by a graphic method, which for all practical purposes is accurate enough; ro and vo are the axes of gears forming angle γ (see diagram, Fig. 11.) On these axes we lay off lines or and ov representing the ratio of the number of teeth (velocity ratio), so that $N_a: N_b::rs:sv$, and



construct parallelogram o r s v. Then, according to McCord,* the angles formed by the tangent s o in the pitch contact o with the axes of the gears insures the least amount of sliding. In bisecting angle γ by tangent u o and using angles produced in this manner we equally distribute the end thrust on both shafts. Both methods have their advantages; to profit by both we select angles α_a and α_b , produced by tangent o x, bisecting angle u o s.

Thus we have when angles are found and C given,

$$P'^{n} = \frac{{}^{2} C \pi \cos \alpha_{a} \cos \alpha_{b}^{e}}{N_{a} \cos \alpha_{b} + N_{b} \cos \alpha_{a}}$$

and when D_a given

$$P'^n = rac{D_a \pi \cos lpha_a}{N_a}$$
 and $D_b = rac{P'^n N_b}{\pi \cos lpha_b}$

^{*} McCord, Kinematics, page 278.

GENERAL FORMULAS.

$$y = \alpha_a + \alpha_b$$

$$P_a^{'n} = P_b^{'n}$$

$$D = \frac{P' N}{\pi} \quad \text{or} = \frac{P'^n N}{\pi \cos \alpha}$$

$$B = D + 2s \quad \text{or} = D - \frac{2}{P^n}$$

$$P' = \frac{D \pi}{N} \quad \text{or} = \frac{P'^n}{\cos \alpha}$$

$$P'^n = P' \cos \alpha$$

$$P'^n = P' \cos \alpha$$

$$P^n = \frac{\pi}{P'^n} \quad \text{(Pitch of cutter.)}$$

$$s = \frac{P'^n}{\pi} \quad \text{or} = \frac{1}{P^n}$$

$$t = \frac{P'^n}{2}$$

$$D'' + f = 2s + \frac{f}{10}$$

$$T = \frac{N}{\cos^2 \alpha} \quad \text{(See Note 1.)}$$

$$L_1 = \frac{N P'}{\tan \alpha} \quad \text{or} \quad \frac{N \pi}{V \tan \alpha} \quad \text{or} \quad \frac{1}{V \ln_n N_b} = \frac{N_b P'_b}{N_b}$$

$$L_2 = \frac{10 \text{ W G}_2}{S \text{ G}_1} \quad \text{(See Note 2 and examples.)}$$

$$\begin{pmatrix} \cos^2 45^2 = .70711 \\ \cos^2 45^2 = .3535 \\ \tan 45^2 = 1.000 \end{pmatrix}$$

Note 1.-Cutters of regular involute system.

Use No.	1 (cutter	for	Τ	from	135 up								21	to	25
• •	2	* *	6.6	* *	• •	55 to	134	• •	6	• •	* *	• •	* 6	17	to	20
6.0	3			٠.		35 to	54	• •	7	• •		. 6	* 4	14	to	16
6.	4	**	• •	• •	• •	26 to	34		5	* *	• •	• •	6.0	12	to	13

Note 2.—Gears used on spiral head and bed for Brown \hat{x} Sharpe milling machine:

W = number of teeth in gear on worm. $G = \cdots \quad \text{if } t \cdots \quad \text{stud.}$ $G_i = \cdots \quad \text{gray} \quad \text{stud.}$

Should a spiral head of different construction be used, the formula might not apply.

The following data are usually required in cutting spiral gears in a Universal Milling Machine, and it will be found convenient to arrange them in tabular form as follows:

(C) •	GEAR.	PINION.
No. of Teeth		
Pitch Diameter		
Outside Diameter		
Circular Pitch		
Angle of Teeth with Axis		
Normal Circular Pitch		
Pitch of Cutter		
Addendum s		
Thickness of Tooth t		
Whole Depth D"+f		
No. of Cutter		
Exact Lead of Spiral		
Approximate Lead of Spiral		
Gears on Milling Machine to Cut Spiral		
Gear on Worm		
ıst Gear on Stud		
2nd Gear on Stud		
Gear on Screw	:	

If the exact lead L_1 can be obtained by the gears at hand, L_1 will equal L_2 and we shall have from the formula

$$L_2 = \frac{\text{Io W } G_2}{\text{S } G_1}$$

$$\frac{L_1}{\text{IO}} = \frac{\text{W } G_2}{\text{S } G_1} \text{ (for B. & S. Milling Machine.)}$$

Example I.

Required the gears for cutting a spiral of 2½" lead.

$$\frac{2\frac{1}{2}}{10} = \frac{I}{4} \text{ factoring, in the most simple way, we have}$$

$$\frac{I}{4} = \frac{I \times I}{2 \times 2} = \frac{I \times 28}{56 \times 2} = \frac{32 \times 28}{56 \times 64} = \frac{W \cdot G_2}{S \cdot G_1}$$

Thus the gearing will be 32 T. on worm, 64 T. 1st. on stud, 28 T. 2nd on stud, and 56 T. on screw.

Trying these gears on the Milling Machine we find that they cannot be used, and as we have no other regular gears in the ratio of 2 to 1 that can be used we must try, by factoring, to get such ratios for the two pairs of gears as to be able to use the gears at hand, bearing in mind that the combined ratio must be \frac{1}{4}.

$$\frac{1}{4} = \frac{18}{72} = \frac{3 \times 6}{9 \times 8} = \frac{24 \times 6}{9 \times 64} = \frac{24 \times 48}{72 \times 64}$$

These gears are at hand and the combination can be used on the machine, giving the exact lead of $2\frac{1}{2}$ ".

Example II.

Required the gears for cutting a spiral of 8.639" lead.

 $8.639 = 8\frac{63.9}{1000}$; reducing, by continued fractions, to a smaller fraction of approximately the same value, as described on pages 74 and 75

Selecting $\frac{16}{25}$ as an approximation near enough for our purpose, and in fact as near as we are likely to find gears for, we have for our lead $8\frac{16}{25}$. Applying the formula as in Example I.

$$\begin{split} \frac{8\frac{16}{25}}{10} &= \frac{W G_2}{S G_1} \\ \frac{8\frac{16}{25}}{10} &= \frac{216}{250} = \frac{108}{125} \text{ factoring we have} \\ \frac{9 \times 12}{25 \times 5} &= \frac{9 \times 48}{100 \times 5} = \frac{72 \times 48}{100 \times 40} \text{ the gears required,} \end{split}$$

these being regular gears furnished with the Milling Machine.

Proof:

$$\frac{72 \times 48 \times 10}{100 \times 40} = 8.640 = L_{2} \\ \underline{8.639} = L_{1} \\ \underline{0.001''} \text{ error in lead.}$$

In shops where much work is done in milling spirals it is desirable to have a full set of gears for the milling machine, from the smallest to the largest numbers of teeth that can be used. This makes it possible, in most cases, to get closer approximations than could be otherwise obtained, and often saves a great deal of figuring.

When the use of continued fractions does not bring a close enough approximation, one method to secure a closer result is to add to or substract from the numerator and denominator of the fraction to be reduced, any numbers nearly in proportion to the given fraction, seeing that the numbers added or substracted are such as to make the fraction reducible to lower terms. By a little ingenuity and patience extremely close approximations can generally be reached in this way.

Take, as an illustration, the fraction in Example II.

$$\frac{8\frac{639}{1000}}{10} = \frac{8639}{10000}$$

Adding 9 to the numerator and 10 to the denominator, these

being in about the same ratio to each other as the numerator and denominator of the fraction, we have

$$\begin{array}{c} 8639 + 9 = 8648 \\ 10000 + 10 = 10010 = \frac{4324}{5005} = \frac{47 \times 92}{55 \times 91} \end{array}$$

All of the gears in this case are special.

Applying the same proof as in Example II. we find that this train of gears will give a lead of 8.6393+, making an error of .0003" in the lead.

No doubt a much closer approximation even than this could be obtained by further trial.

Another method is to multiply both terms of the fraction by some number which will make one term of the fraction easily reducible, and adding one to or subtracting it from the other term to make it possible to reduce that also.

There is an element of uncertainty in both these methods, as we never feel sure that we have obtained the best combination; practical work, however, rarely requires accuracy beyond a point that can readily be reached.

The accompanying list of prime numbers and factors will be found useful in reducing and factoring fractions.

PRIME NUMBERS AND FACTORS. 1 TO 1000.

1 26 2×13 51 3×17 76 $2^2 \times 2^2 \times 13$ 2 2^7 3^3 52 $2^2 \times 13$ 77 7×7 3 2^2 2^3 <th< th=""><th>111 × 13 × 15 × 111 × 7 × 17</th></th<>	111 × 13 × 15 × 111 × 7 × 17
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	×13 45 41 ×7 17
$ \begin{array}{c ccccccccccccccccccccccccccccccccccc$	×7
$ \begin{array}{ c c c c c c c c c c c c c c c c c c c$	×7
$ \begin{array}{ c c c c c c c c c c c c c c c c c c c$	17
$ \begin{array}{ c c c c c c c c c c c c c c c c c c c$	17
$ \begin{array}{ c c c c c c c c c c c c c c c c c c c$	
$\begin{array}{ c c c c c c c c c c c c c c c c c c c$	10
	F3
	29
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	11
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	
$\begin{array}{ c c c c c c c c c c c c c c c c c c c$	$\times 5$
$\begin{array}{ c c c c c c c c c c c c c c c c c c c$	3
	23
$\begin{array}{ c c c c c c c c c c c c c c c c c c c$	31
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	Ł7
$\begin{bmatrix} 20 \end{bmatrix}$ $\begin{bmatrix} 2^2 \times 5 \end{bmatrix}$ $\begin{bmatrix} 45 \end{bmatrix}$ $\begin{bmatrix} 3^2 \times 5 \end{bmatrix}$ $\begin{bmatrix} 70 \end{bmatrix}$ $\begin{bmatrix} 2 \times 5 \times 7 \end{bmatrix}$ $\begin{bmatrix} 95 \end{bmatrix}$ $\begin{bmatrix} 5 \times 1 \end{bmatrix}$	19
$\begin{array}{ c c c c c c c c c c c c c c c c c c c$	3
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	
	72
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	11
	5^2

101		131		161	7 × 23	191	
	0 1 9 1 1 "		02 V 2 V 11				06 v. 0
	$2 \times 3 \times 17$	133					
103	03 v 10					193	
	$2^3 \times 13$				$2^2 \times 41$		i
105	$3 \times 5 \times 7$		$3^3 \times 5$				
106	2×53	1	$2^{\circ} \times 1$		2×83		$2^2 \times 7^2$
107	22 20	137		167		197	
108	$2^2 \times 3^3$	138	$2 \times 3 \times 23$		$2^3 \times 3 \times 7$		$2 \times 3^2 \times 11$
109		139		169	13^{2}	199	
110	$2 \times 5 \times 11$	140	$2^2 \times 5 \times 7$	170	$2 \times 5 \times 17$		$2^{3} \times 5^{2}$
111	3×37	141	3×47	171	$3^2 \times 19$	201	3×67
112	$2^4 \times 7$	142	2×71	172	$2^2 \times 43$	202	2×101
113		143	11×13	173		203	7×29
114	$2 \times 3 \times 19$	144	$2^4 \times 3^2$	174	$2 \times 3 \times 29$	204	$2^2 \times 3 \times 17$
115	5×23	145	5×29	175	$5^2 \times 7$	205	5×41
116	$2^2 \times 29$	146	2×73	176	$2^4 \times 11$	206	2×103
117	$3^2 \times 13$	147	3×7^2	177	3×59	207	$3^{2} \times 23$
118	2×59	148	$2^2 \times 37$	178	2×89	208	$2^4 \times 13$
119	7×17	149		179		209	11×19
120	$2^3 \times 3 \times 5$	150	$2 \times 3 \times 5^2$	180	$2^2 \times 3^2 \times 5$	210	$2 \times 3 \times 5 \times 7$
121	11^{2}	151		181		211	
122	2×61	152	$2^{\circ} \times 19$	182	$2 \times 7 \times 13$	212	$2^2 \times 53$
123	3×41	153	$3^2 \times 17$	183	3×61	213	3×71
124	$2^2 \times 31$	154	$2 \times 7 \times 11$	184	$2^3 \times 23$	214	2×107
125	5^3	155	5×31	185	5×37	215	5×43
126	$2 \times 3^2 \times 7$	156	$2^2 \times 3 \times 13$	186	$2 \times 3 \times 31$	216	$2^{3} \times 3^{3}$
127		157		187	11×17	217	7 × 31
128	27	158	2×79	188	$2^2 \times 47$	218	2×109
129	3×43	159	3×53	189	$3^3 \times 7$	219	3×73
130	$2 \times 5 \times 13$	160	$2^5 \times 5$	190	$2 \times 5 \times 19$	220	$2^2 \times 5 \times 11$

•							
221	13×17	251		281		311	
222	$2 \times 3 \times 37$	252	$2^2 \times 3^2 \times 7$	282	$2 \times 3 \times 47$	312	$2^3 \times 3 \times 13$
223		253	11×23	283		313	
224	$2^5 \times 7$	254	2×127	284	$2^2 \times 71$	314	2×157
225	$3^2 \times 5^2$	255	$3 \times 5 \times 17$	285	$3 \times 5 \times 19$	315	$3^2 \times 5 \times 7$
226	2×113	256	2^8	286	$2 \times 11 \times 13$	316	$2^2 \times 79$
227		257		287	7×41	317	
228	$2^2 \times 3 \times 19$	258	$2 \times 3 \times 43$	288	$2^5 \times 3^2$	318	$2 \times 3 \times 53$
229		259	7×37	289	17^2	319	11×29
230	$2 \times 5 \times 23$	260	$2^2 \times 5 \times 13$	290	$2 \times 5 \times 29$	320	$2^6 \times 5$
231	$3 \times 7 \times 11$	261	$3^2 \times 29$	291	3×97	321	3×107
232	$2^3 \times 29$	262	2×131	292	$2^2 \times 73$	322	$2 \times 7 \times 23$
233		263		293		323	17×19
234	$2 \times 3^2 \times 13$	264	$2^3 \times 3 \times 11$	294	$2 \times 3 \times 7^2$	324	$2^2 \times 3^4$
235	5×47	265	5×53	295	5×59	325	$5^2 \times 13$
236	$2^2 \times 59$	266	$2 \times 7 \times 19$	296	$2^3 \times 37$	326	2×163
237	3×79	267	3×89	297	$3^3 \times 11$	327	3×109
238	$2 \times 7 \times 17$	268	$2^2 \times 67$	298	2×149	328	$2^3 \times 41$
239		269		299	13×23	329	7×47
240	$-2^4 \times 3 \times 5$	270	$2 \times 3^3 \times 5$	300	$2^2 \times 3 \times 5^2$	330	$2\times3\times5\times11$
241		271		301	7×43	331	
242	2×11^2	272	$2^4 \times 17$	302	2×151	332	$2^2 \times 83$
243	\mathfrak{R}^5	273	$3 \times 7 \times 13$	303	3×101	333	$3^2 \times 37$
244	$2^2 \times 61$	274	2×137	304	$2^4 \times 19$	334	2×167
245	5×7^2	275	$5^2 \times 11$	305	5×61	335	5×67
246	$2 \times 3 \times 41$	276	$2^2 \times 3 \times 23$	306	$2 \times 3^2 \times 17$	336	$2^4 \times 3 \times 7$
247	13×19	277		307		337	
248	$2^3 \times 31$	278	2×139	308	$2^2 \times 7 \times 11$	338	2×13^2
249	3×83	279	$3^2 \times 31$	309	3×103	339	3×113
250	2×5^3	280	$2^3 \times 5 \times 7$	310	$2 \times 5 \times 31$	340	$2^2 \times 5 \times 17$

-							
341	11×31	371	7×53	401		431	
342	$2\times3^2\times19$	372	$2^2 \times 3 \times 31$	402	$2 \times 3 \times 67$	432	$2^4 \times 3^3$
343	73	373		403	13×31	433	
344	$2^3 \times 43$	374	$2 \times 11 \times 17$	404	$2^2 \times 101$	434	$2 \times 7 \times 31$
345	$2 \times 5 \times 23$	375	3×5^{3}	405	$3^4 \times 5$	435	$3\times 5\times 29$
346	2×173	376	$2^3 \times 47$	406	$2 \times 7 \times 29$	436	$2^2 \times 109$
347		377	13×19	407	11×37	437	19×23
348	$2^2 \times 3 \times 29$	378	$2 \times 3^{\circ} \times 7$	408	$2^3 \times 3 \times 17$	438	$2 \times 3 \times 73$
349		379		409		439	
350	$2 \times 5^2 \times 7$	380	$2^2 \times 5 \times 19$	410	$2 \times 5 \times 41$	440	$2^3 \times 5 \times 11$
351	$3^3 \times 13$	381	3×127	411	3×137	441	$3^2 \times 7^2$
352	$2^5 \times 11$	382	2×191	412	$2^2 \times 103$	412	$2 \times 13 \times 17$
353		383		413	7×59	443	
354	$2 \times 3 \times 59$	384	$2^7 \times 3$	414	$2 \times 3^2 \times 23$	444	$2^2 \times 3 \times 37$
355	5×71	385	$5 \times 7 \times 11$	415	5×83	445	5×89
356	$2^2 \times 89$	386	2×193	416	$2^{5} \times 13$	446	2×223
357	$3 \times 7 \times 17$	387	$3^2 \times 43$	417	3×139	447	3×149
358	2×179	388	$2^2 \times 97$	418	$2 \times 11 \times 19$	448	$2^6 \times 7$
359		389		419		449	
360	$2^3 \times 3^2 \times 5$	390	$2 \times 3 \times 5 \times 13$	420	$2^2 \times 3 \times 5 \times 7$	450	$2 \times 3^2 \times 5^2$
361	19^{2}	391	17×23	421		451	11×41
362	2×181	392	$2^3 \times 7^2$	422	2×211	452	$2^2 \times 113$
363	3×11^2	393	3×131	423	$3^2 \times 47$	453	3×151
364	$2^2 \times 7 \times 13$	394	2×197	424	$2^3 \times 53$	454	2×227
365	5×73	395	5×79	425	$5^2 \times 17$	455	$5 \times 7 \times 13$
366	$2 \times 3 \times 61$	396	$2^2 \times 3^2 \times 11$	426	$2 \times 3 \times 71$	456	$2^3 \times 3 \times 19$
367		397		427	7×61	457	
368	$2^4 \times 23$	398	2×199	428	$2^2 \times 107$	458	2×229
369	$3^2 \times 41$	399	$3 \times 7 \times 19$	429	$3 \times 11 \times 13$	459	$3^3 \times 17$
370	$2 \times 5 \times 37$	400	$2^4 \times 5^2$	430	$2 \times 5 \times 43$	460	$2^2 \times 5 \times 23$

F							
461		491		521		551	19×29
462	$2\times3\times7\times11$	492	$2^2 \times 3 \times 41$	522	$2 \times 3^2 \times 29$	552	$2^3 \times 3 \times 23$
463		493	17×29	523		553	7×79
464	$2^4 \times 29$	494	$2 \times 13 \times 19$	524	$2^2 \times 131$	554	2×277
465	$3 \times 5 \times 31$	495	$3^2 \times 5 \times 11$	525	$3 \times 5^2 \times 7$	555	$3 \times 5 \times 37$
466	2×233	496	$2^4 \times 31$	526	2×263	556	$2^2 \times 139$
467		497	7×71	527	17×31	557	3
468	$2^2 \times 3^2 \times 13$	498	$2 \times 3 \times 83$	528	$2^4 \times 3 \times 11$	558	$2 \times 3^2 \times 31$
469	7×67	499		529	23^{2}	559	13×43
470	$2 \times 5 \times 47$	500	$2^2 \times 5^3$	530	$2 \times 5 \times 53$	560	$2^4 \times 5 \times 7$
471	3×157	501	3×167	531	$3^2 \times 59$	561	$3 \times 11 \times 17$
472	$2^3 \times 59$	502	2×251	532	$2^2 \times 7 \times 19$	562	2×281
473	11×43	503		533	13 × 41	563	
474	$2 \times 3 \times 79$	504	$2^3 \times 3^2 \times 7$	534	$2 \times 3 \times 89$	564	$2^2 \times 3 \times 47$
475	$5^2 \times 19$	505	5×101	535	5×107	565	5×113
476	$2^2 \times 7 \times 17$	506	$2\times11\times23$	536	$2^{3} \times 67$	566	2×283
477	$3^2 \times 53$	507	3×13^2	537	3×179	567	$3^4 \times 7$
478	2×239	5 08	$2^2 \times 127$	538	2×269	568	$2^3 \times 71$
479		509		539	$7^2 \times 11$	569	
480	$2^5 \times 3 \times 5$	510	$2\times3\times5\times17$	540	$2^2 \times 3^3 \times 5$	570	$2\times3\times5\times19$
481	13×37	511	7×73	541		571	
482	2×241	512	2^9	542	2×271	572	$2^2 \times 11 \times 13$
483	$3 \times 7 \times 23$	513	$3^3 \times 19$	543	3×181	573	3×191
484	$2^2 \times 11^2$	514	2×257	544	$2^5 imes 17$	574	$2 \times 7 \times 41$
485	5×97	515	5×103	545	5×109	575	$5^2 \times 23$
486	2×3^5	516	$2^2 \times 3 \times 43$	546	$2\times3\times7\times13$	576	$2^6 \times 3^2$
487		517	11×47	547		577	
488	$2^3 \times 61$	518	$2 \times 7 \times 37$	548	$2^2 \times 137$	578	2×17^2
489	3×163	519	3×173	549	$3^2 \times 61$	579	3×193
4 90	$2 \times 5 \times 7^2$	520	$2^3 \times 5 \times 13$	550	$2 \times 5^2 \times 11$	580	$2^2 \times 5 \times 29$

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581	7×83	611	13×47	641		671	11×61
582	$2 \times 3 \times 97$	612	$2^2 \times 3^2 \times 17$	642	$2\times3\times107$	672	$2^5 \times 3 \times 7$
583	11×53	613		643		673	
584	$2^{\circ} \times 73$	614	2×307	644	$2^2 \times 7 \times 23$	674	2×337
585	$3^2 \times 5 \times 13$	615	$3 \times 5 \times 41$	645	$3 \times 5 \times 43$	675	$3^{\circ} \times 5^{\circ}$
586	2×293	616	$2^3 \times 7 \times 11$	646	$2\times17\times19$	676	$2^2 \times 13^2$
587		617		647		677	
588	$2^2 \times 3 \times 7^2$	618	$2\times3\times103$	648	$2^{\circ} \times 3^{4}$	678	$2 \times 3 \times 113$
589	19×31	619		649	11×59	679	7×97
590	$2 \times 5 \times 59$	620	$2^2 \times 5 \times 31$	650	$2 \times 5^2 \times 13$	680	$2^3 \times 5 \times 17$
591	3×197	621	$3^3 \times 23$	651	$3 \times 7 \times 31$	681	3×227
592	$2^4 \times 37$	622	2×311	652	$2^2 \times 163$	682	$2 \times 11 \times 31$
593		623	7×89	653		683	
594	$2 \times 3^{\circ} \times 11$	624	$2^4 \times 3 \times 13$	654	$2\times3\times109$	684	$2^2 \times 3^2 \times 19$
595	$5 \times 7 \times 17$	625	$\tilde{\phi}^4$	655	5×131	685	5×137
596	$2^2 \times 149$	626	2×313	656	$2^4 \times 41$	686	2×7^3
597	3×199	627	$3 \times 11 \times 19$	657	$3^2 \times 73$	687	3×229
598	$2 \times 13 \times 23$	628	$2^2 \times 157$	658	$2 \times 7 \times 47$	688	$2^4 \times 43$
5 99		629	17×37	659		689	
600	$2^3 \times 3 \times 5^2$	630	$2\times3^2\times5\times7$	660	$2^2 \times 3 \times 5 \times 11$	690	$2\times3\times5\times23$
601		631		661		691	•
602	$2 \times 7 \times 43$	632	$2^{\circ} \times 79$	662	2×331	692	$2^2 \times 173$
603	$3^2 \times 67$	633	3×211	663	$3 \times 13 \times 17$	693	$3^2 \times 7 \times 11$
604	$2^2 \times 151$	634	2×317	664	$2^{\circ} \times 83$	694	2×347
605	5×11^2	635	5×127	665	$5 \times 7 \times 19$	695	5×139
606	$2 \times 3 \times 101$	636	$2^2 \times 3 \times 53$	666	$2\times3^2\times37$	696	$2^3 \times 3 \times 29$
607		637	$7^2 \times 13$	667	23×29	697	17×41
608	$2^5 \times 19$	638	$2\times11\times29$	668	$2^2 \times 167$	698	2×349
609	$3 \times 7 \times 29$	639	$3^2 \times 71$	669	3×223	699	3×233
610	$2 \times 5 \times 61$	640	$2^7 \times 5$	670	$2 \times 5 \times 67$	700	$2^2 \times 5^2 \times 7$

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701		731	17×43	761		791	7×113
702	$2 \times 3^3 \times 13$	732	$2^2 \times 3 \times 61$	762	$2 \times 3 \times 127$	792	$2^3 \times 3^2 \times 11$
703	19×37	733		763	7×109	793	13×61
704	$2^{6} \times 11$	734	2×367	764	$2^2 \times 191$	794	2×397
705	$3 \times 5 \times 47$	735	$3 \times 5 \times 7^2$	765	$3^2 \times 5 \times 17$	795	$3 \times 5 \times 53$
706	2×353	736	$2^5 \times 23$	766	2×383	796	$2^2 \times 199$
707	7×101	737	11×67	767	13×59	797	
708	$2^2 \times 3 \times 59$	738	$2 \times 3^2 \times 41$	768	$2^8 \times 3$	798	$2\times3\times7\times19$
709		739		769		799	17×47
710	$2 \times 5 \times 71$	740	$2^2 \times 5 \times 37$	770	$2 \times 5 \times 7 \times 11$	800	$2^5 \times 5^2$
711	$3^2 \times 79$	741	$3 \times 13 \times 19$	771	3×257	801	$3^2 \times 89$
712	$2^3 \times 89$	742	$2 \times 7 \times 53$	772	$2^2 \times 193$	802	2×401
713	23×31	743		773		803	11×73
714	$2\times3\times7\times17$	744	$2^3 \times 3 \times 31$	774	$2 \times 3^2 \times 43$	804	$2^2 \times 3 \times 67$
715	$5 \times 11 \times 13$	745	5×149	775	$5^2 \times 31$	805	$5\times 7\times 23$
716	$2^2 \times 179$	746	2×373	776	$2^3 \times 97$	806	$2 \times 13 \times 31$
717	3×239	747	$3^2 \times 83$	777	$3 \times 7 \times 37$	807	3×269
718	2×359	748	$2^2 \times 11 \times 17$	778	2×389	808	$2^3 \times 101$
719		749	7×107	779	19×41	809	
720	$2^4 \times 3^2 \times 5$	750	$2\times3\times5^3$	780	$2^2 \times 3 \times 5 \times 13$	810	$2 \times 3^4 \times 5$
721	7×103	751		781	11×71	811	
722	2×19^2	752	$2^4 \times 47$	782	$2 \times 17 \times 23$	812	$2^2 \times 7 \times 29$
723	3×241	753	3×251	783	$3^3 \times 29$	813	3×271
724	$2^2 \times 181$	754	$2\times13\times29$	784	$2^4 \times 7^2$	814	$2\times11\times37$
725	$5^2 \times 29$	755	-5×151	785	5×157	815	5×163
726	$2 \times 3 \times 11^2$	756	$2^2 \times 3^3 \times 7$	786	$2 \times 3 \times 131$	816	$2^4 \times 3 \times 17$
727		757		787		817	19×43
728	$2^3 \times 7 \times 13$	758	2×379	788	$2^2 \times 197$	818	2×409
729	3^6	759	$3 \times 11 \times 23$	789	3×263	819	$3^2 \times 7 \times 13$
730	$2 \times 5 \times 73$	760	$2^3 \times 5 \times 19$	790	$2 \times 5 \times 79$	820	$2^2 \times 5 \times 41$
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821		851	23×37	881		911	
822	$2 \times 3 \times 137$	852	$2^2 \times 3 \times 71$	882	$2 \times 3^2 \times 7^2$	912	$2^4 \times 3 \times 19$
823		853		883		913	11×83
824	$2^{3} \times 103$	854	$2 \times 7 \times 61$	884	$2^2 \times 13 \times 17$	914	2×457
825	$3 \times 5^2 \times 11$	855	$3^2 \times 5 \times 19$	885	$3 \times 5 \times 59$	915	$3 \times 5 \times 61$
826	$2 \times 7 \times 59$	856	$2^3 \times 107$	886	2×443	916	$2^2 \times 229$
827		857		887		917	7×131
828	$2^2 \times 3^2 \times 23$	858	$2 \times 3 \times 11 \times 13$	888	$2^3 \times 3 \times 37$	918	$2 \times 3^3 \times 17$
829		859		889	7×127	919	
830	$2 \times 5 \times 83$	860	$2^2 \times 5 \times 43$	890	$2 \times 5 \times 89$	920	$2^3 \times 5 \times 23$
831	3×277	861	$3 \times 7 \times 41$	891	$3^4 \times 11$	921	3×307
832	$2^6 \times 13$	862	2×431	892	$2^2 \times 223$	922	2×461
833	$7^2 \times 17$	863		893	19×47	923	13×71
834	$2 \times 3 \times 139$	864	$2^5 \times 3^3$	894	$2 \times 3 \times 149$	924	$2^2 \times 3 \times 7 \times 11$
835	5×167	865	5×173	895	5×179	925	$5^2 \times 37$
836	$2^2 \times 11 \times 19$	866	2×433	896	$2^7 \times 7$	926	2×463
837	$3^3 \times 31$	867	3×17^2	897	$3 \times 13 \times 23$	927	$3^2 \times 103$
838	2×419	868	$2^2 \times 7 \times 31$	898	2×449	928	$2^5 \times 29$
839		869	11×79	899	29×31	929	
840	$2^3 \times 3 \times 5 \times 7$	870	$2\times3\times5\times29$	900	$2^2 \times 3^2 \times 5^2$	930	2×3×5×31
841	29^{2}	871	13×67	901	17×53	931	$7^2 \times 19$
842	2×421	872	$2^{\circ} \times 109$	902	$2 \times 11 \times 41$	932	$2^2 \times 233$
843	3×281	873	$3^2 \times 97$	903	$3 \times 7 \times 43$	933	3×311
844	$2^2 \times 211$	874	$2\times19\times23$	904	$2^{3} \times 113$	934	2×467
845	5×13^2	875	$5^3 \times 7$	905	5×181	935	$5 \times 11 \times 17$
846	$2 \times 3^2 \times 47$	876	$2^2 \times 3 \times 73$	906	$2 \times 3 \times 151$	936	$2^3 \times 3^2 \times 13$
847	7×11^2	877		907		937	
848	$2^4 \times 53$	878	. 2×439	908	$2^2 \times 227$	938	$2 \times 7 \times 67$
849	3×283	879	3×293	909	$3^2 \times 101$	939	3×313
850	$2 \times 5^2 \times 17$	880	$2^4 \times 5 \times 11$	910	$2\times5\times7\times13$	940	$2^2 \times 5 \times 47$

941		956	$2^2 \times 239$	971		986	$2 \times 17 \times 29$
942	$2 \times 3 \times 157$		$3 \times 11 \times 29$	1			$3 \times 7 \times 47$
943	23×41	958	2×479	973	7×139	988	$2^2 \times 13 \times 19$
944	$2^4 \times 59$	959	7×137	974	2×487	989	23×43
945	$3^3 \times 5 \times 7$	960	$2^6 \times 3 \times 5$	975	$3 \times 5^2 \times 13$	990	$2\times3^2\times5\times11$
946	$2 \times 11 \times 43$	961	31^{2}	976	$2^4 \times 61$	991	
947		962	$2 \times 13 \times 37$	977		992	$2^5 \times 31$
948	$2^2 \times 3 \times 79$	963	$3^2 \times 107$	978	$2 \times 3 \times 163$	993	3×331
949	13×73	964	$2^2 \times 241$	979	11×89	994	$2 \times 7 \times 71$
950	$2 \times 5^2 \times 19$	965	5×193	980	$2^2 \times 5 \times 7^2$	995	5×199
951	3×317	966	$2 \times 3 \times 7 \times 23$	981	$3^2 \times 109$	996	$2^2 \times 3 \times 83$
952	$2^3 \times 7 \times 17$	$\boldsymbol{967}$		982	2×491	997	
953		968	$2^3 \times 11^2$	983		998	2×499
954	$2 \times 3^2 \times 53$	969	$3 \times 17 \times 19$	984	$2^3 \times 3 \times 41$	999	$3^3 \times 37$
955	5×191	970	$2 \times 5 \times 97$	985	5×197	1000	$2^3 \times 5^3$

CHAPTER VI.

INTERNAL GEARING.

PART A.-INTERNAL SPUR GEARING.

(Figs. 12, 13, 14, 15, 16.)

A little consideration will show that a tooth of an internal or annular gear is the same as the space of a spur—external gear.

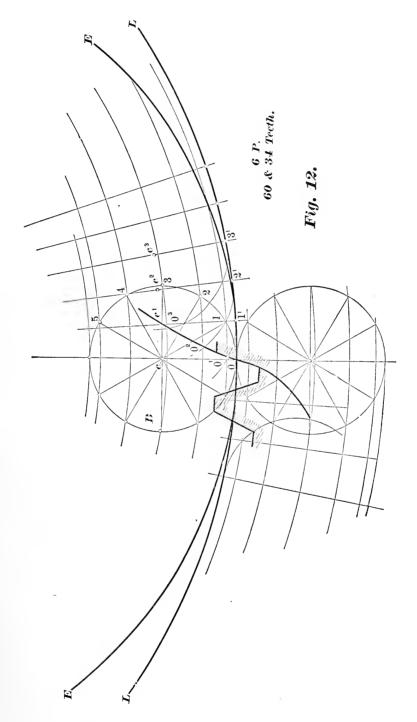
We prefer the epicycloidal form of tooth in this class of gearing to the involute form, for the reason that the difficulties in overcoming the interference of gear teeth in the involute system are considerable. Special constructions are required when the difference between the number of teeth in gear and pinion is small.

In using the system of epicycloidal form of tooth in which the gear of 15 teeth has radial flanks, this difference must be at least 15 teeth, if the teeth have both faces and flanks. Gears fulfilling this condition present no difficulties. Their pitch diameters are found as in regular spur gears, and the inside diameter is equal to the pitch diameter, less twice the addendum.

If, however, this difference is less than 15, say 6, or 2, or 1, then we may construct the tooth outline (based on the epicycloidal system) in two different ways.

First Method.—To explain this method better, let us suppose the case as in Fig. 12, in which the difference between gear and pinion is more than 15 teeth. Here the point o of the describing circle B (the diameter of which in the best practice of the present day is equal to the pitch radius of a 15 tooth gear, of the same pitch as the gears in question) generates the cycloid o, o¹, o², o³, etc., when rolling on pitch circle L L of gear, forming the face of tooth; and when rolling on the outside of L L the flank of the tooth. In like manner is the face and flank of the pinion tooth produced by B rolling outside and inside of E E (pitch circle of pinion). A little study

of Fig. 12 (in which the face and flank of a gear tooth are produced) will show the describing circle B divided into 12



equal parts and circles laid through these points (1, 2, 3, etc.), concentric with L L. We now lay off on L L the distances 0-1, 1-2, 2-3, etc., of the circumference of B, and obtain points

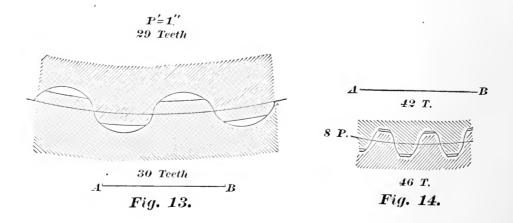
 I^1 , I^2 , I^3 , etc. [Ordinarily it is sufficient to use the chord.] It will now readily be seen that B in rolling on L L will successively come in contact with I^1 , I^2 , I^3 , etc., I^2 meanwhile moving to I^2 , I^3 , etc. (points on radii through I^1 , I^2 , I^3 , etc.), and the generating point o advancing to I^3 , I^3 , etc., being the intersections of B with I^3 , I^4 , I^4 , I^4 , I^5 , etc., as centers and the circles laid through I^2 , I^3 , etc. Points I^4 , I^4 ,

In this manner the form of tooth is obtained, when the difference of teeth in gear and pinion is less than 15, with the exception that the diameter of describing circle B

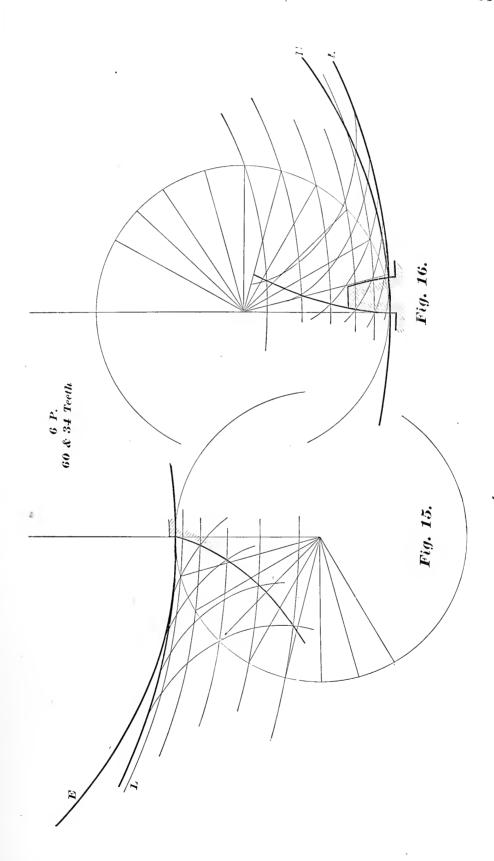
$$= \frac{1}{2} \left(\frac{N - n}{P} \right)$$

where P = diametral pitch, N and n number of teeth in gears.

The distances of the tooth above and below the pitch line as well as the thickness t are determined as in regular spur gears by the pitch, except when the difference in gear and pinion is very small, where we obtain a short tooth, as in Figs. 13 and 14. In such a case the height of tooth is arbitrary and only conditioned by the curve. In internal gears it is best to allow more clearance at bottom of tooth than in ordinary spur gears.



In a construction of this kind it is suggested to draw the tooth outline many times full size and reduce by photography. An equally multiplied line A B will help in reducing.



Second Method.—The difference between gear and pinion being very small, it is sometimes desirable to obtain a smooth action by avoiding what is termed the "friction of approaching action."* This is done, the pinion driving, by giving gear only flanks, Fig. 15, and the gear driving, by giving gear only faces, Fig. 16. In both these cases we have but one describing circle, whose diameter is equal to the difference of the two pitch diameters. The construction of the curve is precisely the same as described under A. The describing circle has been divided into 24 parts simply for the sake of greater accuracy.

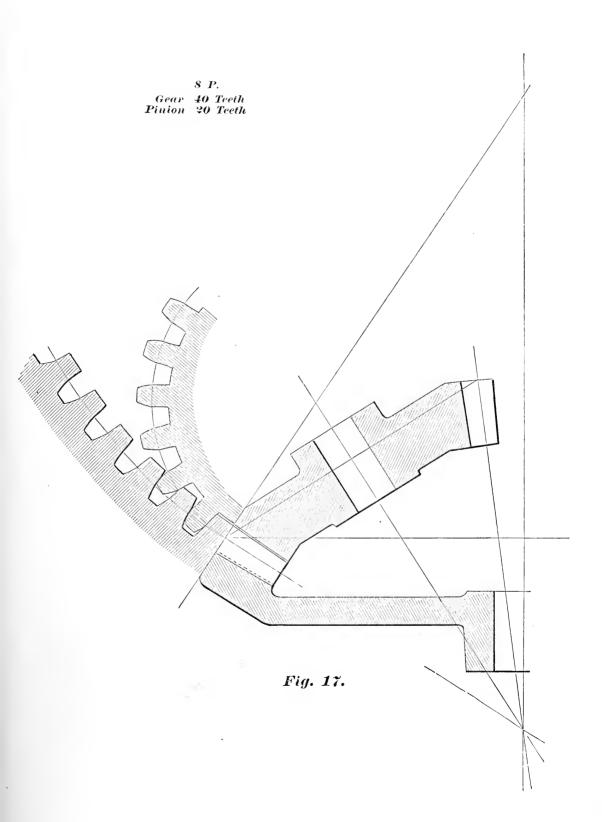
PART B.-INTERNAL BEVEL GEARS.

(Fig. 17.)

The pitch surfaces of bevel gears are cones whose apexes are at a common point, rolling upon each other. The tooth forms for any given pair of bevel gears are the same as for a pair of spur gears (of same pitch) whose pitch radii are equal to the respective apex distances of the normal cones (i. e., cones whose elements are perpendicular upon the elements of the bevel gear pitch cones). (Compare Fig 19, page 68.)

The same is true of internal bevel gears, with the modification that here one of the pitch cones rolls inside of the other. The spur gears to whose tooth forms the forms of the bevel gear teeth correspond, resolve themselves into internal spur gears (Fig. 17). The problem is now to be solved as indicated in the first part of this chapter.

[#] McCord, Kinematics, pages 107, 108.



CHAPTER VII.

GEAR PATTERNS.

(Fig. 18.)

To place in bevel gears the best iron where it belongs, the tooth side of the pattern should always be in the nowel, no matter of what shape the hubs are.

Hubs, if short, may be left solid on web; if long they should be made loose. A long hub should go on a tapering arbor, to prevent tipping in the sand. 1° taper for draft on hubs when loose, and 3° when solid is considered sufficient.

Coreprints as a rule are made separate, partly to allow the pattern to be turned on an arbor, partly for convenience, should it be desirable to use different sizes.

Put rap- and draw-holes as near to center as possible. Referring to Fig. 18, make L = D for D from $\frac{3}{4}$ " to $\frac{1}{2}$ ", or even more, should hubs be very long. Otherwise if D is more than $\frac{1}{2}$ " leave $L = \frac{1}{2}$ ".

Iron pattern before using should be marked, rusted and waxed.

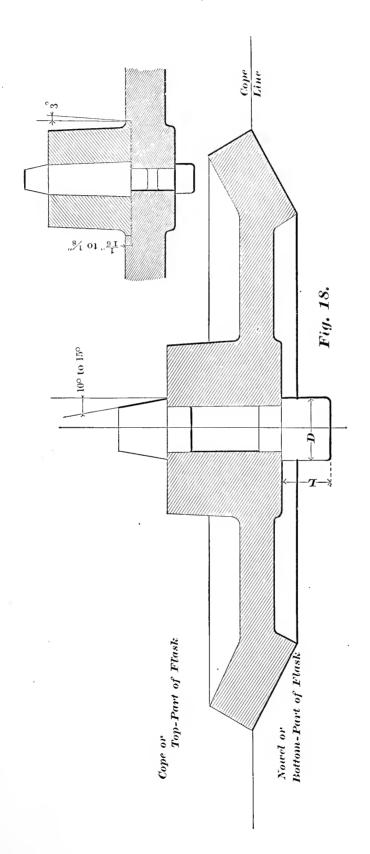
Shrinkage—For cast-iron,
$$\frac{1}{8}$$
" per foot.
For brass, $\frac{3}{16}$ " "

Cast-iron gears, especially arm gears, do not always shrink 1/8" per foot. In making iron patterns the following allowances have been found useful:

Up to 12" diameter allow no shrink.

From 12" to 18" " " ½ regular shrink.
" 18" to 24" " " ½ " "
" 24" to 48" " " ½3 " "
Above 48" " .10" " "

for cast-iron.



If in gears the teeth are to be cast, the tooth thickness t in the pattern is made smaller than called for by the pitch, to avoid binding of the teeth when cast. No definite rule can be given, as the practice varies on this point. For the different diametral pitches we would advise making t smaller by an amount expressed in inches, as given in the following table:

DIAM. PITCH.	AMOUNT t IS SMALLER.	DIAM, РІТСН.	AMOUNT t	
16	.010"	5	.020′′	
12	.012"	4	.022"	
10	.014"	3	.026"	
8	.016''	2	.030"	
6	.018"	Ĭ	.040"	

CHAPTER VIII.

DIMENSIONS AND FORM FOR BEVEL GEAR CUTTERS.

(Fig. 19.)

The data needed to determine the form and thickness of a bevel gear cutter are the following:

P = pitch.

N = number of teeth in large gear.

n = number of teeth in small gear.

F = length of face of tooth, measured on pitch line.

After having laid out a diagram of the pitch cones a b c and a b f, and laid off the width of face, the problem resolves itself into two parts:

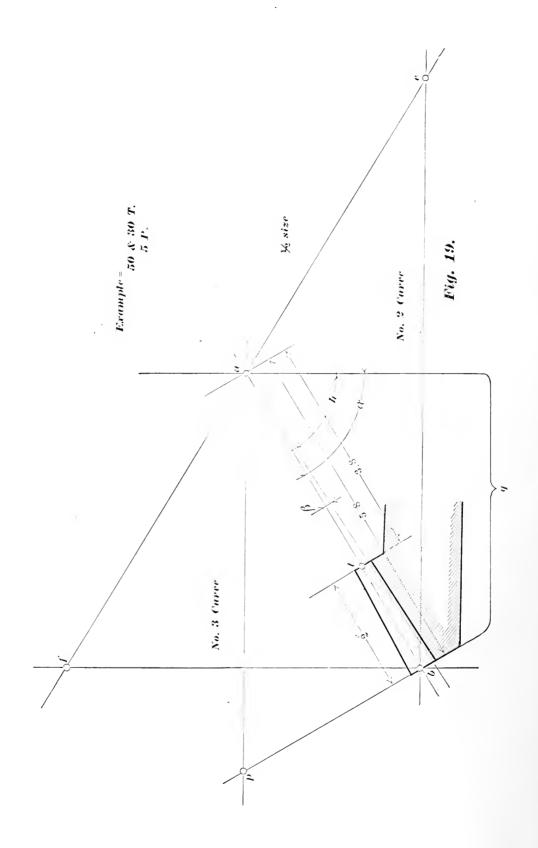
PART I.—DETERMINE PROPER CURVE FOR CUTTER.

It will be remembered that in the involute system of cutters (the only one used for bevel gears that are cut with rotary cutter), a set of eight different cutters is made for each pitch, numbering from No. 1 to No. 8, and cutting from a rack to 12 teeth. Each number represents the form of a cutter suitable to cut the indicated number of teeth. For instance, No. 4 cutter (No. 4 curve) will cut 26 to 34 teeth. In order to find the curve to be used for gear and pinion we simply construct the normal pitch cones by erecting the perpendicular p q through p, Fig. 19. We now measure the lines p p and p p, and taking them as radii, multiplying each by 2 and P we obtain a number of teeth for which cutters of proper curves may be selected. From example we have:

Gear:
$$b \ q = 9\frac{3}{4}$$
"; $2 \times P \times 9.75 = 97 \text{ T}$ No. 2 curve.
Pinion: $b \ p = 3\frac{1}{2}$ "; $2 \times P \times 3.5 = 35 \text{ T}$ No. 3 curve.

The eight cutters which are made in the involute system for each pitch are as follows:

No.	I	will cut	wheels	from	135	teeth	to	a ra	ick.
"	2	"	"	"	5.5	66	"	134	teeth.
"	3	"	. 6	"	35		"		"
"	4	"	"	"	26	"	"	34	"
"	5	"	"	"	2 I	"	"	25	66
"	6	"	"	"	17	"	"	20	"
"	7	"	"	"	14	"	"	16	"
"	8	"	66	"	12	"	"	13	"



PART II. - DETERMINE THICKNESS OF CUTTER.

It is very evident that a bevel gear cutter cannot be thicker than the width of the space at small end of tooth; the practice is to make cutter .oo5" thinner. Theoretically the cutting angle (h) is equal to pitch angle less angle of bottom (or $h = \alpha - \beta'$). Practically, however, better results are obtained by making $h = \alpha - \beta$ (substituting angle of top for angle of bottom), and in calculating the depth at small end, to add the full clearance (f) to the obtained working depth, giving equal amount of clearance at large and small end. This is done to obtain a tooth thinner at the top and more curved. As the small end of tooth determines the thickness of cutter, we shall have to find the tooth part values at small end. From the diagram it will be seen that the values at large end are to those at small end as their respective apex distances (a b and a l). numerical values of these can be taken from the diagram and the quotient of the larger in the smaller is the constant wherewith to multiply the tooth values at large end, to obtain those at small end. In our example we find:

$$a \ l = 3.8 \\ a \ b = 5.8 = .655 = \text{constant}$$
 For 5 P we have:
 $t = .3141$ $t' = .2057$
 $s = .2000$ $s' = .1310$
 $f = .0314$ $f = .0314$
 $s + f = .2314$ $s' + f = .1624$
 $D'' + f = .4314$. $s' = .1310$
 $D''' + f = .2934$

From the foregoing it is evident that a spur gear cutter could not be used, since a bevel gear cutter must be thinner.

If in gears of more than 30 teeth the faces are proportionately long, we select a cutter whose curve corresponds to the midway section of the tooth. The curve of the cutter is found by the method explained in Part I. of this Chapter.

CHAPTER IX.

DIRECTIONS FOR CUTTING BEVEL GEARS WITH ROTARY CUTTER.

(Fig. 20.)

In order to obtain good results, the gear blanks must be of the right size and form. The following sizes for each end of the tooth must be given the workman:

Total depth of tooth.

Thickness of tooth at pitch line.

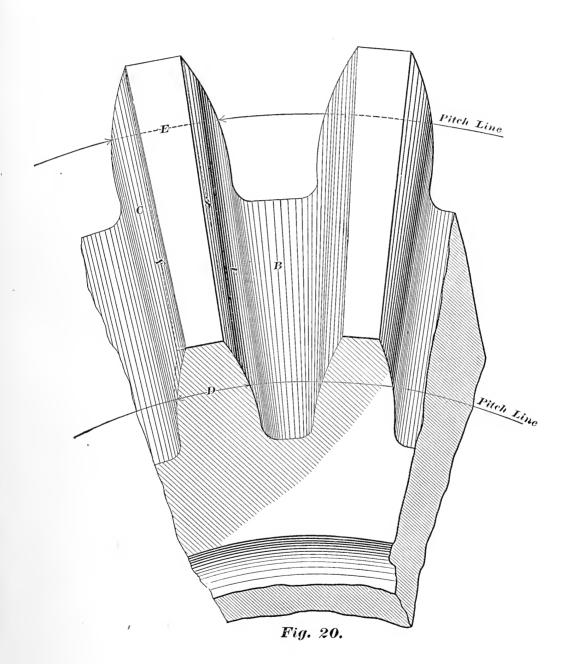
Height of tooth above pitch line.

These sizes are obtained as explained in Chapter VIII.

The workman must further know the cutting angle (see formula on page 13 and compare Chapter VIII.), and be provided with the proper tools with which to measure teeth, etc.

In cutting a gear on a universal milling machine the operations and adjustments of the machine are as follows:

- 1. Set spiral bed to zero line.
- 2. Set cutter central with spiral head spindle.
- 3. Set spiral head to the proper cutting angle.
- 4. Set the index on head for the number of teeth to be cut, leaving the sector on the straight or numbered row of holes, and set the pointer (or in some machines the dial) on cross-feed screw of milling machine to zero line.
- 5. As a matter of precaution, mark the depth to be cut for large and small end of tooth on their respective places.
- 6. Cut two or three teeth in blank to conform with these marks in depth. The teeth will now be too thick on both their pitch circles.
- 7. Set the cutter off the center by moving the saddle to or from the frame of the machine by means of the cross-feed screw, measuring the advance on dial of same. The saddle must not be moved further than what to good judgment



appears as not excessive; at the same time bearing in mind that an equal amount of stock is to be taken off each side of tooth.

- 8. Rotate the gear in the opposite direction from which the saddle is moved off the center, and trim the sides of teeth (A) (Fig. 20.)
- 9. Then move the saddle the same distance on the opposite side of center and rotate the gear an equal amount in the opposite direction and trim the other sides of teeth (C).
- ro. If the teeth are still too thick at large end E, move the saddle further off the center and repeat the operation, bearing in mind that the gear must be rotated and the saddle moved an equal amount each way from their respective zero settings.

It is generally necessary to file the sides of teeth above the pitch line more or less on the small ends of teeth, as indicated by dotted lines F F. This applies to pinions of less than 30 teeth.

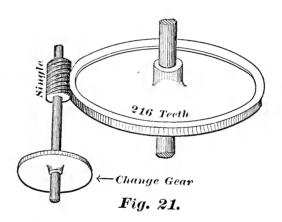
For gears of coarser pitch than 5 diametral it is best to make one cut around before attempting to obtain the tooth thickness.

The formulas for obtaining the dimensions and angles of gear blanks are given in Chapter III.

CHAPTER X.

THE INDEXING OF ANY WHOLE OR FRACTIONAL NUMBER.

(Fig. 21)



In indexing on a machine the question simply is: How many divisions of the machine index have to be advanced to advance a unit division of the number required. To which is the

$$answer = \frac{divisions of machine index}{number to be indexed}$$

Suppose the number of divisions in index wheel of machine to be 216.

EXAMPLE I.—Index 72.

Answer:
$$\frac{216}{7^2} = 3$$
 (3 turns of worm).

EXAMPLE II.—Index 123.

$$\frac{216}{123} = 1 + \frac{93}{123}$$

If now we should put on worm shaft a change gear having 123 teeth, give the worm shaft, Fig. 21, one turn, and in addition thereto advance 93 teeth of the change gear (to give the fractional turn), we would have indexed correctly one unit of the given number, and so solved the problem. Should we not have change gear 123 we may try those on hand. The question then is: How many teeth (χ) of the gear on hand (for instance 82) must we advance to obtain a result equal to the one when advancing 93 teeth of the 123 tooth gear? We have:

$$\frac{93}{123} = \frac{\chi}{82}$$
 where $\chi = 62$

Example III.—Index 365, change gear 147.

$$\frac{216}{365} = \frac{\chi}{147}$$
 where $\chi = 87 - \frac{3}{365}$

Here 147 is the change gear on hand. In indexing for a unit of 365 we advance87 teeth of our 147 tooth gear. It is evident that in so doing we advance too fast and will have indexed three teeth of our change gear too many when the circle is completed. To avoid having this error show in its total amount between the last and the first division, we can distribute the error by dropping one tooth at a time at three even intervals.

Example IV.—Index 190.

$$\frac{216}{190} = 1 + \frac{26}{190}$$
 Change gear on hand 88 T
 $\frac{26}{190} = \frac{\chi}{88}$ where $\chi = 12 + \frac{8}{190}$

To distribute the error in this case we advance one additional tooth ot a time of the change gear at eight even intervals.

Example V.—Index 117.3913.

$$\frac{216}{117.3913} = 1 + \frac{986087}{1173913}$$

This example is in nowise different from the preceding ones, except that the fraction is expressed in large numbers. This fraction we can reduce to lower approximate values, which for practical purposes are accurate enough. This is done by the method of continued fractions. [For an explana-

tion of this method we refer to our "Practical Treatise on Gearing."]

$$\frac{986087}{1173913}$$
986087) 1173913 (1
$$\frac{986087}{187826}) 986087 (5
$$\frac{939130}{46957}) 187826 (3
\frac{140871}{46955}) 46957 (1
\frac{46955}{2}) 46955 (23477
\frac{46954}{1}) 2 (2
\frac{2}{0}$$

$$\frac{986087}{1173913} = \frac{1}{1+\frac{1}{5+\frac{1}{2}}}
\frac{1}{3+\frac{1}{1}}
\frac{1}{1+\frac{1}{23477+\frac{1}{2}}}$$

$$\frac{1}{a=1} \quad b=5 \quad d=\frac{16}{19} \quad \frac{21}{25} \quad \frac{493033}{586944} \quad \frac{986087}{1173913}$$$$

Note.—Find the first two fractions by reduction $\frac{1}{1} = \frac{1}{1}$ and $\frac{1}{1+\frac{1}{5}} = \frac{5}{6}$; the

others are then found by the rule $\begin{cases} b & c + a = d \\ b^1 & c + a^1 = d^1 \end{cases}$

The fraction $\frac{2}{2}\frac{1}{5}$ is a good approximation; putting therefore a change gear of 25 teeth on worm shaft, we advance (beside the one full turn) 21 teeth to index our unit.

Of course, in using any but the correct fraction we have an error every time we index a division; so that when indexed around the whole circle, we have multiplied this error by the number of divisions.

In the present example this error is evidently equal to the difference between the correct and the approximate fraction used. Reducing both common fractions to decimal fractions we have:

$$\frac{986087}{1173913} = .84000006$$

$$\frac{21}{25} = .84000000$$

$$\frac{.00000006}{.00000006} = \text{error in each division.}$$

.00000006 X 117.3913 = .00000704348 total error in complete circle. This error is expressed in parts of a unit division. (To find this error expressed in inches, multiply it by the distance between two divisions, measured on the circle.) In this case the approximate fraction being smaller than the correct one, in indexing the whole circle we fall short .00000704348 of a division.

Example VI.—Index 15.708

$$\frac{216}{15.708} = 13 + \frac{11796}{15708}$$

$$\frac{11796}{15708} = \frac{983}{1309}$$

$$983) \frac{1309}{326} (1)$$

$$\frac{983}{326}) 983 (3)$$

$$\frac{978}{5}) 326 (65)$$

$$\frac{25}{1}) 5 (5)$$

$$\frac{5}{5}$$

$$\frac{983}{1309} = \frac{1}{1 + \frac{1}{5}}$$

$$\frac{1}{65 + \frac{1}{5}}$$

$$\frac{1}{1} \frac{3}{4} \frac{196}{261} \frac{983}{1309}$$

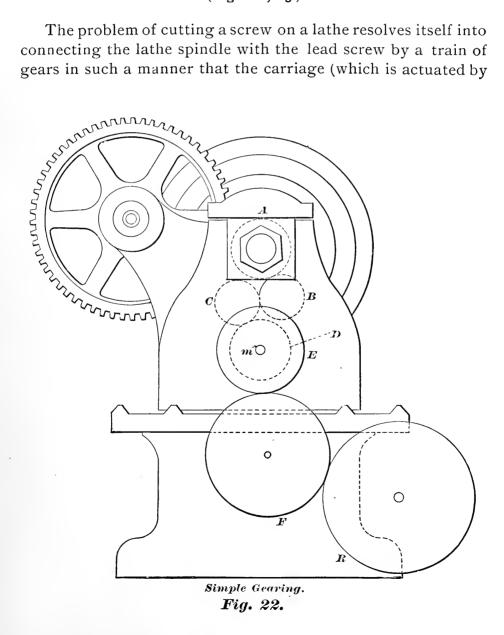
In using the approximation $\frac{196}{261}$ the error for each division (found as above) will be .00002927, for the whole circle .0000460. In this case, the approximation being larger than the correct fraction, we overreach the circle by the error.

CHAPTER XI.

THE GEARING OF LATHES FOR SCREW CUTTING.

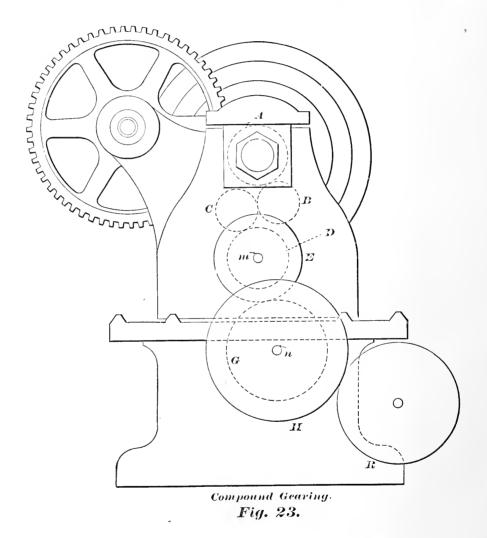
(Figs. 22, 23.)

The problem of cutting a screw on a lathe resolves itself into connecting the lathe spindle with the lead screw by a train of gears in such a manner that the carriage (which is actuated by



the lead screw) advances just one inch, or some definite distance, while the lathe spindle makes a number of revolutions equal to the number of threads to be cut per inch.

The lead screw has, with the exception of a very few cases, always a single thread, and to advance the carriage one inch it therefore makes a number of revolutions equal to its number



of threads per inch. Should the lead screw have double thread, it will, to accomplish the same result, make a number of revolutions equal to half its number of threads per inch. It follows that we must know in the first place the number of threads per inch on lead screw.

It ought to be clearly understood that one or more intermediate gears, which simply transmit the motion received from one gear to another, in no wise alter the ultimate ratio of a train of gearing. An even number of intermediate gears simply change the direction of rotation, an odd number do not alter it.

The gearing of a lathe to solve a problem in screw cutting can be accomplished by

A. Simple gearing.

B. Compound gearing.

Referring to the diagrams, Figs. 22 and 23, we have in Fig. 22 a case of simple, and in Fig. 23 a case of compound gearing.

In simple gearing the motion from gear E is transmitted either directly to gear R on lead screw or through the intermediate F. In compound gearing the motion of E is transmitted through two gears (G and H) keyed together, revolving on the same stud n, by which we can change the velocity ratio of the motion while transmitting it from E to R. With these four variables E, G, H, R, we are enabled to have a wider range of changes than in simple gearing.

B and C, being intermediate gears, are not to be considered. If, as is generally the case, gear A equals gear D, we disregard them both, simply remembering that gear E (being fast on same shaft with D) makes as many revolutions as the spindle. Sometimes gear D is twice as large as gear A, then, still considering gear E as making as many revolutions as the spindle, we deal with the lead screw as having twice as many threads per inch as it measures.

SIMPLE GEARING.

Let there be: the number of teeth in the different gears expressed by their respective letters, as per Fig. 22, and

s = threads per inch to be cut, L = threads per inch on lead screw; then

If now one of the two gears E and R is selected, the other will be:

$$R = \frac{s E}{L}$$
; $E = \frac{L R}{s}$

2. The two gears may be found by making

$$R = p s$$

 $E = p L$ where p may be any number.

3. The above holds good when a fractional thread is to be cut, but if the fraction is expressed in large numbers, as, for instance, $s = 2.833 \ (2\frac{833}{1000})$, we first reduce this fraction $(\frac{833}{1000})$ to lower approximate values by the process of continued fraction (see pages 73 and 74).

833)
$$\frac{1000}{1000}$$
 (1
 $\frac{833}{1000}$) 833 (4
 $\frac{668}{1000}$) $\frac{167}{2}$ (1
 $\frac{165}{2}$) $\frac{165}{5}$ (82
 $\frac{16}{5}$) $\frac{4}{10}$ 2 (2
 $\frac{2}{0}$)
 $\frac{1}{10}$ $\frac{4}{5}$ $\frac{1}{5}$ $\frac{82}{6}$ $\frac{2}{414}$ $\frac{833}{1000}$
 $\frac{5}{0}$ = .833 (nearly) and $s = 2\frac{5}{6}$

If in this case L = 4, and we select E = 48, then, since

$$R = \frac{s E}{L} \quad R = 34$$

COMPOUND GEARING.

4. In a lathe geared compound for cutting a screw the product of the drivers (E and H. Fig. 23) multiplied by the number of threads per inch to be cut must equal the product of the driven (G and I multiplied by the number of threads on lead screw. This is expressed by

E , H ,
$$s = G$$
 , R , L or $\frac{E \cdot H \cdot s}{G \cdot R \cdot L} = r$

If three of the gears E, H, G, R have been selected, the fourth one would be either

$$E = \frac{G R L}{H s} \quad \text{or}$$

$$H = \frac{G R L}{E s} \quad \text{or}$$

$$G = \frac{E H s}{R L} \quad \text{or}$$

$$R = \frac{E H s}{G L}$$

$$s = \frac{R G L}{E H} = L \left(\frac{R \cdot G}{L \cdot E \cdot H}\right)$$

If a fractional thread is to be cut, as under "3," we reduce the fraction to lower approximate values.

Example.—Gear for 5.2327 threads per inch, lead screw is 6 threads.

$$.2327 = \frac{2327}{10000}$$

$$2327) \frac{2000}{10000} (4)$$

$$\frac{9308}{692}) \frac{2327}{2327} (3)$$

$$\frac{2076}{251}) \frac{692}{692} (2)$$

$$\frac{502}{190}) \frac{251}{251} (1)$$

$$\frac{190}{61}) \frac{190}{61} (8)$$

$$\frac{5}{2}) \frac{5}{2} (2)$$

$$\frac{4}{1}) \frac{3}{2} (2)$$

$$\frac{2}{2}$$

$$\frac{1}{4} \frac{3}{13} \frac{7}{30} \frac{10}{43} \frac{37}{159} \frac{306}{1315} \frac{343}{1474} \frac{992}{4263} \frac{2327}{10000}$$

$$\frac{10}{43} = .2327 \text{ (nearly) and } 5.2327 = 5\frac{10}{43}$$
Selecting E = 43, H = 52, R = 50, and
$$G = \frac{E \cdot H \cdot s}{R \cdot L} \text{ we have } G = \frac{43 \cdot 52 \cdot 5\frac{10}{43}}{50 \cdot 6} = 39.$$

5. The examples so far given all deal with single thread. The pitch of a screw is the distance from center of one thread to the center of the next. The lead of a screw is the advance for each complete revolution. In a single thread screw the pitch is equal to the lead, while in a double thread screw the pitch is equal to one-half the lead; in a triple thread screw equal to one-third the lead, etc.

If we have to gear a lathe for a many-threaded screw (double, triple, quadruple, etc.), we simply ascertain the lead, and deal with the lead as we would with the pitch in a single thread screw, i. e., we divide one inch by it, to obtain the number of threads for which we have to gear our lathe.

Example.—Gear for double thread screw, lead = .4654. Number of threads per inch to be geared for is:

$$\frac{1}{\text{Lead}} = \frac{1}{.4654} = 2.1.;87$$

Lead screw is four threads per inch.

As in previous examples, we reduce the fraction .1487= $\frac{1487}{10000}$ to lower approximate values by the process of continued fraction.

From the different values received in the usual way we select:

$$\frac{1}{14}$$
 = .1487 (nearly) and 2.1487 = $2\frac{1}{14}$

We have therefore:

$$s = 2\frac{11}{14}$$

$$L = 4$$

$$\begin{cases} E = 74 \\ G = 30 \\ H = 40 \end{cases}$$

$$R = \frac{E \cdot H \cdot s}{G \cdot L} = \frac{74 \cdot 40 \cdot 2\frac{11}{14}}{30 \cdot 4} = 53$$

Note.--In using any but the original fraction we commit an error. This error can be found by reducing the approximate fraction used to a decimal fraction, and comparing it with the original fraction. In the above example the original fraction is

.1487 and
$$\frac{\frac{1}{74} = .74864}{Error = .0006}$$
 inch in lead.

In cutting a multiple screw, after having cut one thread, the question arises how to move the thread tool the correct amount for cutting the next thread.

In cutting double, triple, etc., threads, if in simple or compound gearing the number of teeth in gear E is divisible by 2, 3, etc., we so divide the teeth; then leaving the carriage at rest we bring gear E out of mesh and move it forward one division, whereby the spindle will assume the correct position.

When E is not divisible we find how many turns (V) of gear R are made to each full turn of the spindle. Dividing this number by 2 for double, by 3 for triple thread, etc., we advance R so many turns and fractions of a turn, being careful to leave the spindle at rest.

For compound gearing:

$$V = \frac{E \cdot H}{G \cdot R}$$

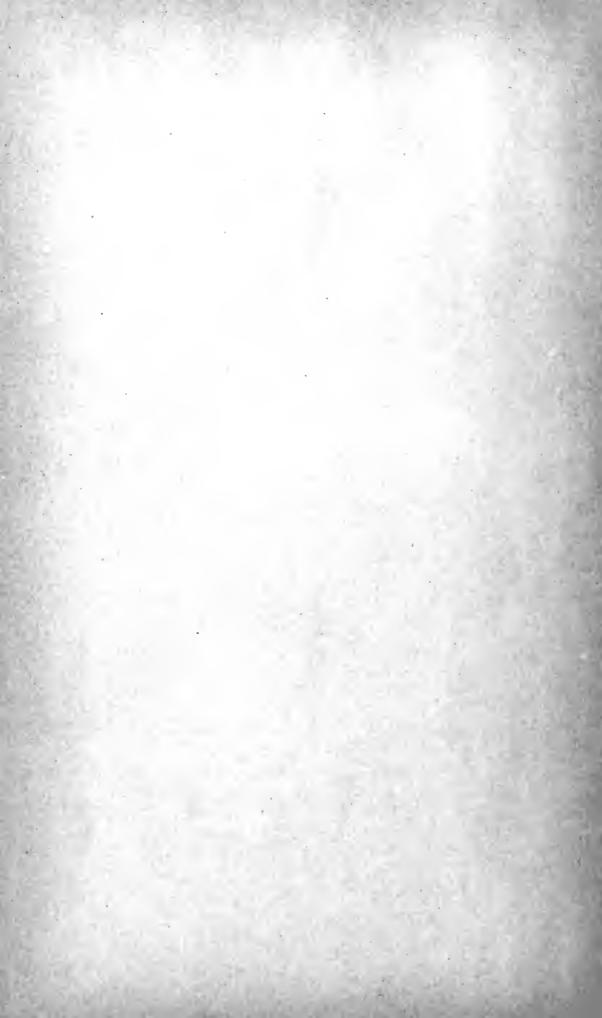
When the gear D is twice as large as the gear A (as explained in fifth paragraph, page 78.) the formula would be

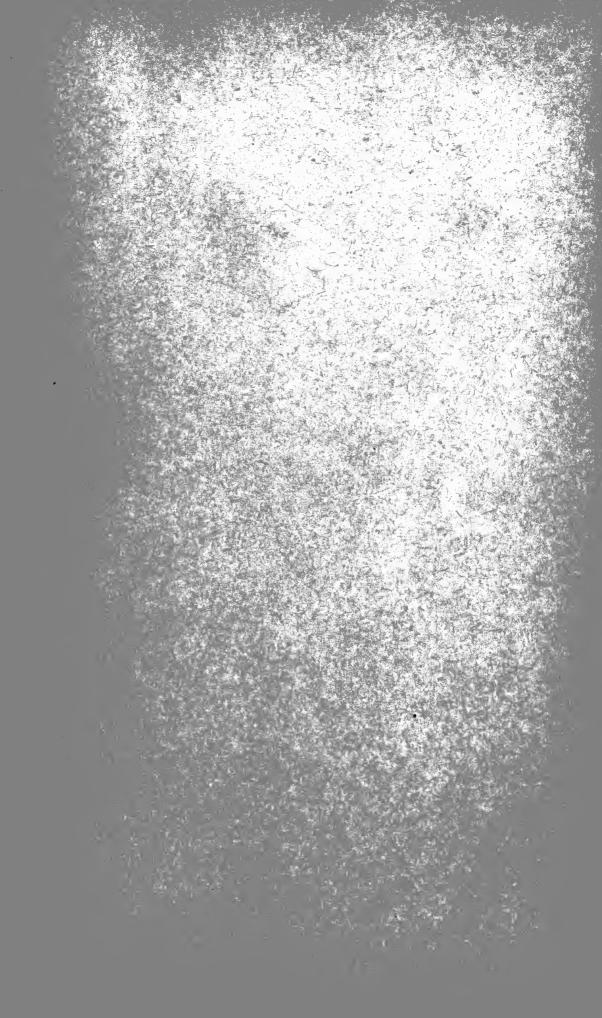
$$V = \frac{E. H.}{2 G. R.}$$

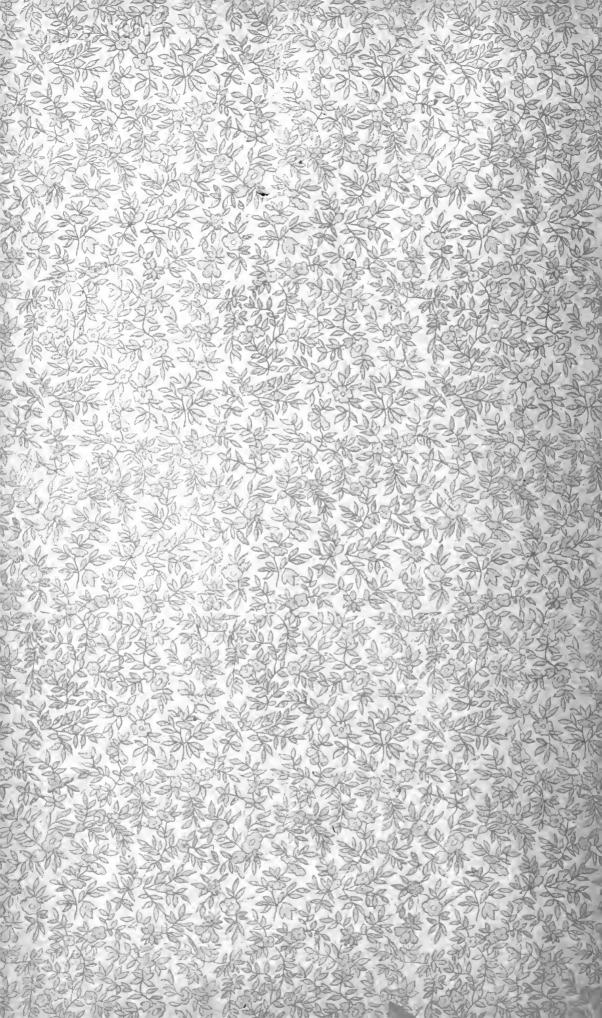
If in simple gearing both E and R are not divisible, one remedy would be to gear the lathe compound; or the face-plate may be accurately divided in two, three or more slots, and all that is then necessary is to move the dog from one slot to another, the carriage remaining stationary.

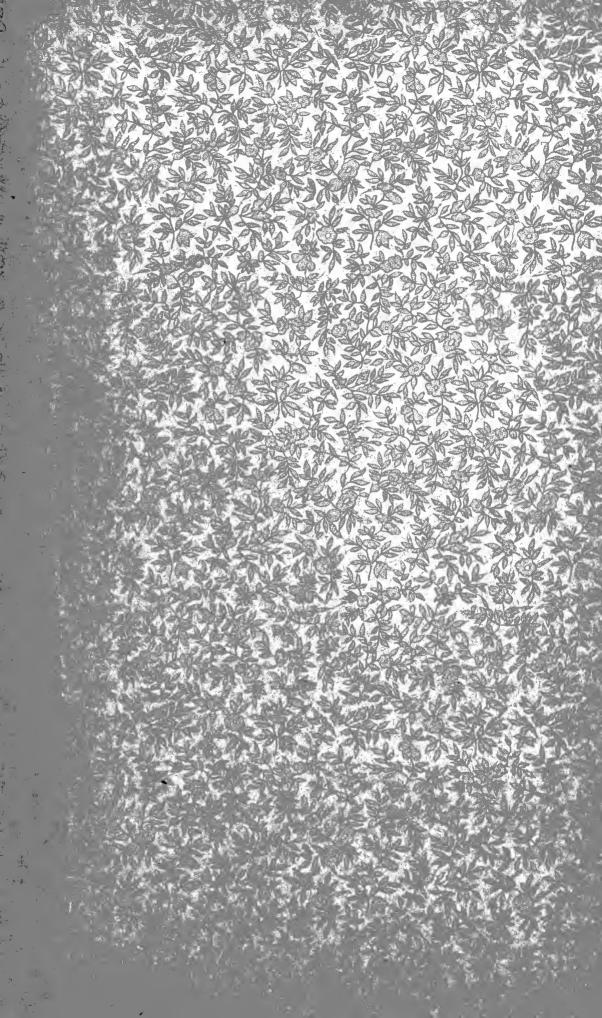
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